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Complex multicomponent real-fluid thermodynamic model for high-pressure Diesel fuel injection

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Abstract

The Perturbed-Chain Statistical Associating Fluid Theory (PC-SAFT) has been coupled with Vapor-Liquid Equilibrium (VLE) calculations in a density-based solver of the Navier-Stokes equations to perform multicomponent two-phase simulations of Diesel injections at high-pressure conditions. This molecular-based EoS requires three empirically determined but well-known parameters to model the properties of a specific component, and thus, there is no need for extensive model calibration, as is typically the case when the NIST (REFPROP) library is utilised. PC-SAFT can handle flexibly the thermodynamic properties of multi-component mixtures for which the NIST (REFPROP) library supports only limited component combinations. Moreover, complex hydrocarbon mixtures can be modelled as a single pseudo-component knowing its number averaged molecular weight (MW) and hydrogen-to-carbon (HN/CN) ratio. Published molecular dynamic simulations have been utilised to demonstrate that the developed algorithm properly captures the VLE interface at high-pressure conditions. Several advection test cases and shock tube problems were performed to validate the numerical framework using analytical and exact solutions. Additionally, two-dimensional simulations of n-dodecane and Diesel injections into nitrogen are included to demonstrate the multidimensional, multispecies and multiphase capability of the numerical framework.

Keywords: Subcritical, PC-SAFT EoS, Diesel Fuel Injection

Nomenclature

List of abbreviations

AAD	Average Absolute Deviation
CFD	Computational Fluid Dynamics
CFL	Courant–Friedrichs–Lewy
CPA	Cubic Plus Association
ECN	Engine Combustion Network
ENO	Essentially Non-Oscillatory
EoS	Equation of State
FC	Fully Conservative
GC	Group Contribution

46	HLLC	Harten-Lax-van Leer-Contact
47	HN/CN	Hydrogen-to-Carbon Ratio
48	LES	Large Eddy Simulation
49	MW	Number Averaged Molecular Weight
50	N-S	Navier-Stokes
51	PNAs	Poly-Nuclear Aromatics
52	PR	Peng-Robinson
53	PC-SAFT	Perturbed Chain Statistical Associating Fluid Theory
54	QC	Quasi-Conservative
55	RK2	Second-order Runge–Kutta
56	SAFT-BACK	Boublik-Alder-Chen-Kreglewshi
57	SRK	Soave-Redlich-Kwong
58	SSP-RK3	Third-order strong-stability-preserving Runge–Kutta
59	TPn flash	Isothermal-Isobaric Flash
60	TVD	Total Variation Diminishing
61	TPD	Tangent Plane Distance
62	VLE	Vapor-Liquid Equilibrium
63	WENO	Weighted Essentially Non-Oscillatory

64

65 *List of Symbols*

66	$\tilde{\alpha}^{res}$	Reduced Helmholtz free energy [-]
67	c	Sound speed [m s^{-1}]
68	d	Temperature-dependent segment diameter [\AA]
69	e	Internal energy [J mol^{-1}]
70	h	Enthalpy [J mol^{-1}]
71	g	Gibbs energy [J mol^{-1}]
72	I	Integrals of the perturbation theory [-]
73	k_b	Boltzmann constant [J K^{-1}]
74	m	Number of segments per chain [-]
75	\bar{m}	Mean segment number in the system [-]
76	M_M	Molecular weight [g/mol]
77	N_A	Avogadro's number [mol^{-1}]
78	p	Pressure [Pa]
79	p_c	Critical pressure [Pa]
80	R	Gas constant [$\text{J mol}^{-1} \text{K}^{-1}$]
81	T	Temperature [K]
82	T_c	Critical temperature [K]
83	x_i	Mole fraction of component i [-]
84	w	Acentric factor [-]
85	Z	Compressibility factor [-]

86

87 *Greek Letters*

88	β	Overall fraction of vapour phase [-]
89	ε	Depth of pair potential [J]
90	η	Packing fraction [-]
91	ρ	Density [kg/m^3]
92	ρ_m	Total number density of molecules [$1/\text{\AA}^3$]

93	σ_d	Segment diameter [\AA]
94	θ	Vapour volume fraction [-]
95	φ	Fugacity coefficient [-]
96	μ	Chemical potential [J mol^{-1}]
97		
98	<i>Superscripts</i>	
99	<i>EQ</i>	Equilibrium
100	<i>disp</i>	Contribution due to dispersive attraction
101	<i>hc</i>	Residual contribution of hard-chain system
102	<i>hs</i>	Residual contribution of hard-sphere system
103	<i>id</i>	Ideal gas contribution
104		

105 1. Introduction

106 This research is focused on improving the thermodynamic models employed to
107 simulate fuel mixing at elevated pressures. To correctly model the combustion in Diesel engines
108 one needs to characterise the atomisation and mixing of sprays. However, even nowadays these
109 processes are not completely understood. According to the experiments performed by several
110 authors [1]–[5], supercritical mixing exists at pressures near or slightly higher than the critical
111 pressure of the liquid fuel. [6], [7] stated that the convection-diffusion phenomena described by
112 the Navier-Stokes equations governs Diesel engine conditions. More recently, in [8] it was
113 determined that the surface tension remains in effect at the gas-liquid interfaces in ambient
114 conditions slightly above the critical point of the fuel. However, at higher pressure and
115 temperature conditions the surface tension diminishes, as expected for supercritical fuel-air
116 mixtures. Diesel engine operation conditions are considered to be in the diffused controlled
117 mixing regime. In a following study [9], the authors carried out systematic measurements using
118 high-speed long-distance microscopy for three single-component fuels (n-heptane, n-
119 dodecane, n-hexadecane) injected into gas (89.71% N_2 , 6.52% CO_2 and 3.77% H_2O) at elevated
120 temperatures (700–1200 K) and pressures (2–11 MPa). The classical evaporation/diffusive
121 mixing boundaries were moved towards higher pressures and temperatures placing Diesel
122 engines conditions in the classical evaporation regime. In [10] the evaporation of n-alkane fuels
123 into nitrogen was investigated at different pressure and temperature conditions carrying out
124 molecular dynamic simulations. The aim of this work was to understand how the transition
125 from classical two-phase evaporation to one-phase diffusion-controlled mixing takes place.
126 Two regimes are identified: (1) subcritical evaporation where a distinctive interface exists
127 separating the liquid core and the ambient gases; and (2) supercritical evaporation where
128 initially the liquid has a surface tension that decreases rapidly and vanishes. In the supercritical
129 evaporation regime, the evaporation rate increases and reaches a maximum after which there is
130 a transition to the supercritical stage. The results obtained have a high degree of agreement
131 against the experimental results obtained by [9].

132 Numerous simulations of Diesel sprays in the literature exist, which employ
133 Lagrangian methods considering a sharp gas-liquid interface which evolve according to
134 primary and secondary breakup models and evaporation [11]–[13]. However, this configuration
135 presents some limitations to accurately capture dense flow regimes near the nozzle where the
136 liquid fuels disintegrate into ligaments that then form droplets. Moreover, they are sensitive to
137 calibration parameters. In [14], [15] an Eulerian density-based methodology was used to model
138 the primary atomisation of the injected liquid accounting for compressibility effects associated
139 with the high-pressure and injection velocity. A single-phase dense-gas approach was combined

140 with the Peng-Robinson (PR) EoS. However, n-dodecane/nitrogen mixtures are a TYPE IV
141 mixture, which means that the critical temperature of the mixture is higher than the lower
142 critical temperature of the components and lower than then the higher critical temperature of
143 the compounds. On the other hand, the critical pressure is higher than the critical pressure of
144 the components. Considering that the pressures that can be found in the combustion chamber
145 of Diesel engines are lower than the critical pressure of some nitrogen/fuel mixtures, the VLE
146 state must be included in the simulation. In [16], [17] a multi-species two-phase model for
147 Eulerian large-eddy simulations (LES) was developed. A thermodynamic solver that can
148 compute the properties of a homogenous mixture in supercritical or subcritical states was
149 employed. The LES including VLE thermodynamics of the so-called Spray A benchmark case
150 of the Engine Combustion Network (ECN) performed by [16] shows a high degree of
151 agreement against the available experimental data. Although according to [9], [10] the Spray A
152 ambient conditions (900K, 6MPa) fall in the classic evaporation regime, the authors of [16]
153 justified the use of a diffuse interface due to the high Weber number and low Stokes number.
154 The authors pointed out the issues of employing cubic EoS for modelling hydrocarbon
155 properties at temperatures found inside the injection system [18]–[20]. In [16], due to the 8.6%
156 error when compared to NIST (REFPROP) in the density prediction of n-dodecane at 363K, it
157 was necessary to increase the injection velocity to match the mass-flow measurement leading
158 to an error in the predicted velocity of 50 m/s. These problems could be overcome by applying
159 SAFT models.

160 The SAFT EoS is based on the perturbation theory, as extensively studied in [21]–[24].
161 This EoS was developed by [25], [26] applying Wertheim's theory and extending it to mixtures.
162 Each molecule of the mixture is decomposed into spherical segments of equal size forming a
163 repulsive, hard sphere reference fluid. The attractive interactions between segments are
164 included in the model as well as the segment-segment energy needed to form a chain between
165 the hard-sphere fluid segments. If the segments exhibit associative interactions such as
166 hydrogen bonding, a term for this interaction is also included. Among the different variants of
167 the SAFT model, the PC-SAFT is the one implemented here. In this model, hard chains are
168 used as the reference fluid instead of hard spheres. While the SAFT EoS computes segment-
169 segment attractive interactions, the PC-SAFT EoS computes chain-chain interactions, which
170 improves the thermodynamic description of chain-like, fluid mixtures [27]. This molecular-
171 based EoS only requires three empirically determined parameters (when the association term is
172 neglected) to model the properties of a specific component without the need for extensive model
173 calibration. Several publications have highlighted the advantages of the SAFT variants with
174 respect to cubic EoS. [28] shows how the PC-SAFT model presents better results than cubic
175 EoS predicting gas phase compressibility factors and oil phase compressibility. For example,
176 Average Absolute Deviation (AAD) of the gas compressibility factors in the range $P=0-1000$
177 bar and $T=0-250$ °C for nC_6 are 0.0144 for PC-SAFT, 0.0479 for SRK (applying the Peneloux
178 volume correction) and 0.0425 for PR (applying the Peneloux volume correction). For nC_5 , they
179 are 0.0127, 0.0529 and 0.0296 respectively. [29] indicated that the PC-SAFT EoS shows a
180 superior performance to the Cubic Plus Association (CPA) EoS in correlating second order
181 derivative properties, such as speed of sound, dP/dV and dP/dT derivatives, heat capacities and
182 the Joule–Thomson coefficient in the alkanes investigated. The CPA model presents a
183 diverging behaviour in the speed of sound attributed to the wrong description of the dP/dV
184 derivative. Similarly, [30] points out the superiority of the SAFT-BACK (Boublik-Alder-Chen-
185 Kreglewski) model over the PR EOS. The SAFT-BACK EoS shows reasonable results for the
186 speed of sound in the vapor and liquid phases (AAD% = 2.3%, 2.1%, and 1.8% for methane,
187 ethane, and propane, respectively). However, the results obtained by PR EOS are only similar

188 to measured data at low pressure conditions. The predicted values at high density ranges present
189 an AAD% for methane, ethane, and propane of 28.6%, 14.7%, and 61.2%, respectively.
190 Moreover, in [31] it was shown how cubic EoS predict a linear increase of the Z factor
191 (compressibility factor) with pressure, while the PC-SAFT EoS presents pressure dependence.

192 In [19], the PC-SAFT was used to close the Navier-Stokes equations using both a
193 conservative and a quasi-conservative formulation, where the double flux model of [32]–[34]
194 was applied. It was observed how the conservative formulation generates spurious pressure
195 oscillations while the quasi-conservative scheme presents an error in the energy conservation
196 that produce an unphysical quick heat-up of the fuel. In [18], supercritical injections of Diesel
197 fuel modelled as surrogates comprising four, five, eight and nine components were performed
198 taking advantage of the capacity of the PC-SAFT EoS to flexibly handle the thermodynamic
199 properties of multi-component mixtures. Simulations at affordable CPU times were carried out
200 by reducing the number of times the PC-SAFT EoS is solved by computing the pressure and
201 sonic fluid velocity in the cell centres and performing a reconstruction of these variables at each
202 cell face. This technique was found to smooth-out the spurious pressure oscillations associated
203 with conservative schemes when used along with real-fluid EoS. The novelty in this paper is to
204 present a numerical framework that combines PC-SAFT and VLE calculations in a density-
205 based, fully conservative solver of the Navier-Stokes and energy conservation equations. VLE
206 calculations allow to perform simulations where the fuel enters the combustion chamber at low
207 temperatures (subcritical injections). Published molecular dynamic simulations have been
208 employed to demonstrate that the algorithm properly captures the multicomponent VLE
209 interface at high-pressure conditions. A purely predictive method that employs the PC-SAFT
210 EoS for developing pseudo-components, which are defined to replicate the properties of
211 complex hydrocarbon mixtures (e.g., diesel fuels), has been completed and validated to be used
212 in CFD simulations. Then, complex hydrocarbon mixtures can be modelled as a single pseudo-
213 component knowing its MW and HN/CN ratio. Advection test cases and shock tube problems
214 were performed to validate the numerical framework. Two-dimensional simulations of planar
215 Diesel jets are performed to demonstrate the capability of the developed methodology to model
216 subcritical mixing at high-pressure conditions.

217

218 **2. PC-SAFT Theory and Methodology**

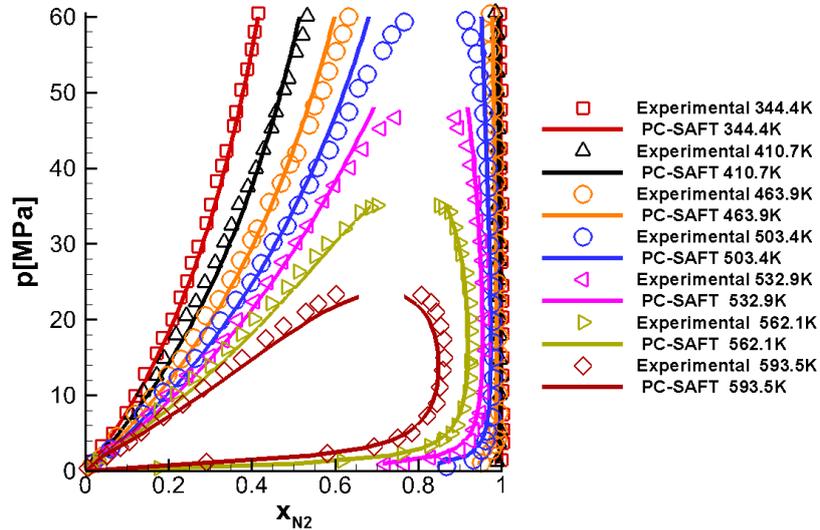
219 In this section it is explained the numerical methodology employed to couple the Navier-Stokes
220 equations, total energy equation, VLE calculations and PC-SAFT model in the same numerical
221 framework. The results of the molecular model and VLE calculations were validated using the
222 experimental results of [45], see Figure 1.

223

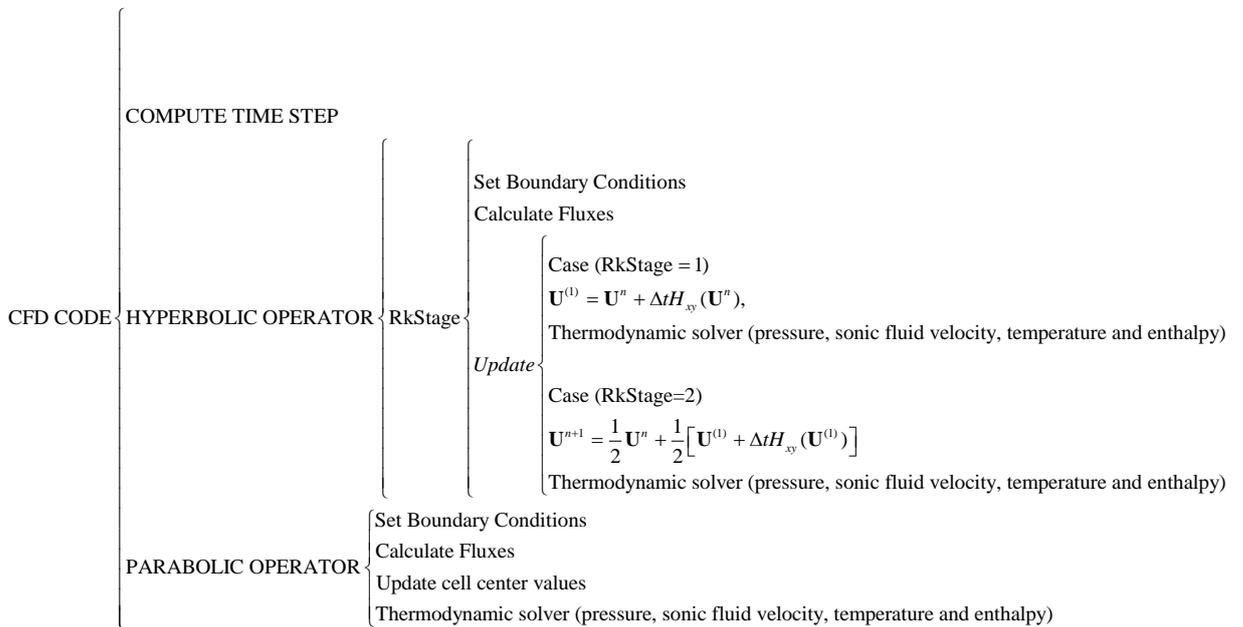
224 **2.a. CFD code**

225 The Navier-Stokes equations for a non-reacting multi-component mixture containing N species
226 in a x-y 2D Cartesian system have been solved employing the finite volume method. Operator
227 splitting as described in [35] is utilised to separate the hyperbolic and parabolic operators. The
228 global time step is computed using the CFL (Courant-Friedrichs-Lewy) criterion of the
229 hyperbolic part. A thermodynamic solver inspired by the work of [16] is employed to
230 approximate the mixture thermophysical properties by performing PC-SAFT and VLE
231 calculations. To compute the convective fluxes: the conservative variables, pressure and speed
232 of sound are interpolated at the cell faces from cell centres using a fifth-order WENO (Weighted
233 Essentially Non-Oscillatory) scheme [18]; the multicomponent HLLC (Harten-Lax-van Leer-
234 Contact) solver is applied to solve the Riemann problem [36]; and the temporal integration is

235 carried out using a second-order Runge–Kutta (RK2) method applying the filter presented in
 236 [18]. In the parabolic sub-step, linear interpolation is performed for computing the conservative
 237 variables, temperature and enthalpy on the cell faces from the corresponding values at the cell
 238 centres. The model developed by [37] is used to calculate the dynamic viscosity and the thermal
 239 conductivity. Figure 2 shows a schematic representation of the CFD code. See Appendix 1.
 240



241
 242 **Figure 1. Experimental [38] and calculated pressure-composition phase diagram for the N₂(1) +**
 243 **C₁₂H₂₆(2) system. Solid lines: PC-SAFT EoS with $k_{ij} = 0.144$**
 244



245
 246 **Figure 2. Schematic representation of the CFD code**
 247

248
 249 **2.b. Diesel modelling**

250 Two approximations have been considered to model the properties of Diesel.

251
 252
 253

254 **Multicomponent Diesel surrogates**

255 In [39] four Diesel surrogates have been proposed, divided into two types depending how close
 256 their composition is to real Diesel. The V0A and V0B are two low-accuracy surrogates while
 257 V1 and V2 are the two higher-accuracy surrogates. Their molar composition is summarized in
 258 Table 1. As pointed out in [40], the PC-SAFT EoS shows the highest degree of agreement with
 259 the experimental values in comparison with the results obtained using the model developed at
 260 NIST [39], see Table 2.

261
 262 **Pseudo-component method**

263 In [41] was developed a technique that defines a single pseudo-component to represent the
 264 compounds found in a hydrocarbon mixture. It only requires two mixture properties as inputs,
 265 the MW and HN/CN ratio. Here we briefly describe how to achieve the pseudo-component PC-
 266 SAFT parameters needed in this study. The group contribution (GC) parameters of [42] are
 267 used to develop the correlations shown in Table 4 for n-alkanes and poly-nuclear aromatics
 268 (PNAs) that numerically bound the pseudo-component PC-SAFT parameter values. An
 269 averaging parameter, Z, is used to calculate the pseudo-component parameters using Eqs. 1-3.
 270 Eqs 4-7 show that Z is calculated using the mixture MW and HN/CN ratio, which can be
 271 directly calculated knowing the mixture components or can be obtained using elemental
 272 analysis for unknown mixtures. Considering that the PC-SAFT is implemented using loops that
 273 depend on the number of components solved, this method allows us to model complex
 274 hydrocarbon mixtures as one component, thus, reducing significantly the computational
 275 requirements of the simulation but with increasing its accuracy.

276
 277 **Table 1. PC-SAFT pure component parameters [40], [42]**

Compound	m	σ (Å)	ϵ/k (K)
n-hexadecane	6.669	3.944	253.59
n-octadecane	7.438	3.948	254.90
n-eicosane	8.207	3.952	255.96
heptamethylnonane	5.603	4.164	266.46
2-methylheptadecane	7.374	3.959	254.83
n-butylcyclohexane	3.682	4.036	282.41
1,3,5-triisopropylcyclohexane	4.959	4.177	297.48
trans-decalin	3.291	4.067	307.98
perhydrophenanthrene	4.211	3.851	337.52
1,2,4-trimethylbenzene	3.610	3.749	284.25
1,3,5-triisopropylbenzene	5.178	4.029	296.68
tetralin	3.088	3.996	337.46
1-methylnaphthalene	3.422	3.901	337.14

278
 279
 280 **Table 2 Comparison between experimentally measured surrogate densities (kg/m³) at 293.15 K**
 281 **and 0.1 MPa with the NIST and PC-SAFT predictions**

Surrogate	Experiment[39]	NIST	PC-SAFT
V0A	818.0	809.1	814.9
V0B	837.5	821.6	833.2
V1	828.4	814.1	825.2
V2	853.0	839.9	861.8

282

283 The methodology developed by [41] was validated for modelling density, isothermal
 284 compressibility and volumetric thermal expansion coefficient of hydrocarbon mixtures, jet and
 285 diesel fuels. However, the pseudo-component must correctly model the internal energy
 286 (employed in the conservation of the total energy equation) speed of sound (used to calculate
 287 the hyperbolic fluxes and time step), enthalpy (employed in the parabolic operator of the
 288 Navier-Stokes equations) and fugacity coefficients (to perform VLE calculations). Using the
 289 PC-SAFT, the internal energy, enthalpy and heat capacities at constant pressure and volume
 290 (needed to compute the speed of sound) are computed as the sum of ideal and residual
 291 contributions. The PC-SAFT pure component parameters obtained employing the method of
 292 [41] are used to calculate the residual contributions. The ideal enthalpy of each component is
 293 calculated by integrating the ideal heat capacity at constant pressure with respect to temperature
 294 [43]. The molar composition of the mixture is used to calculate the ideal enthalpy of the
 295 mixture. The ideal internal energy of the mixture is computed employing the ideal enthalpy of
 296 the mixture. The ideal heat capacities at constant pressure of each component is computed
 297 employing the correlations published in [43]; then, molar fractions are used to compute the
 298 ideal heat capacity at constant pressure of the mixture, which is employed to calculate the ideal
 299 heat capacity at constant volume.

300
 301

Table 3. Molar composition for the four Diesel fuel surrogates (V0A, V0B, V1, V2) [39]

Compound	V0A	V0B	V1	V2
n-hexadecane	27.8	-	2.70	-
n-octadecane	-	23.5	20.2	10.8
n-eicosane	-	-	-	0.80
heptamethylnonane	36.3	27.0	29.2	-
2-methylheptadecane	-	-	-	7.3
n-butylcyclohexane	-	-	5.10	19.1
triisopropylcyclohexane	-	-	-	11.0
trans-decalin	14.8	-	5.50	-
perhydrophenanthrene	-	-	-	6.00
1,2,4-trimethylbenzene	-	12.5	7.5	-
1,3,5-triisopropylbenzene	-	-	-	14.7
tetralin	-	20.9	15.4	16.4
1-methylnaphthalene	21.1	16.1	14.4	13.9

302

303

$$304 \quad m_{\text{pseudo-component}} = (1 - Z)m_{\text{n-alkane}} + Zm_{\text{PNA}} \quad (1)$$

305

$$306 \quad (m\sigma)_{\text{pseudo-component}} = (1 - Z)(m\sigma)_{\text{n-alkane}} + Z(m\sigma)_{\text{PNA}} \quad (2)$$

308

$$307 \quad \left(\frac{\varepsilon}{k}\right)_{\text{pseudo-component}} = (1 - Z)\left(\frac{\varepsilon}{k}\right)_{\text{n-alkane}} + Z\left(\frac{\varepsilon}{k}\right)_{\text{PNA}} \quad (3)$$

309

310

$$311 \quad Z = \begin{cases} \frac{\text{DoU}_{\text{mixture}}}{\text{DoU}_{\text{PNA}}}, & \text{MW}_{\text{mixture}} < 178 \text{ g/mol} \\ \frac{\text{DoU}_{\text{mixture}}}{10}, & \text{MW}_{\text{mixture}} \geq 178 \text{ g/mol} \end{cases} \quad (4)$$

312

$$313 \quad \text{DoU}_{\text{PNA}} = 0.05993 \times \text{MW} - 0.68158 \quad (5)$$

314

$$315 \quad CN_{\text{mixture}} = \frac{MW_{\text{mixture}}}{12.01 + 1.01((HN/CN)_{\text{mixture}})} \quad (6)$$

316

$$317 \quad DoU_{\text{mixture}} = \frac{1}{2}(2 \times CN_{\text{mixture}} + 2 - HN_{\text{mixture}}) \quad (7)$$

318

319

320 **Table 4. PC-SAFT parameter correlations as a function of MW (g/mol) for n-alkanes and PNAs**
 321 **using the GC parameters of [42]**

	n-alkane	PNA
m	$0.0274MW + 0.4648$	$0.0163MW + 0.9256$
$m\sigma$ (Å)	$0.1092MW + 1.5677$	$0.0612MW + 3.5324$
ε/k (K)	$\exp(5.5811 - 10.2507/MW)$	$\exp(5.5657 - 8.6620/MW)$

322

323

324 **2.c. Thermodynamic solver (PC-SAFT + VLE)**

325 The thermodynamic solver is employed to compute temperature, pressure, sound speed and
 326 enthalpy once the conservative variables have been updated. The inputs are the density, internal
 327 energy and mass fraction of the components. Three pure component parameters per compound
 328 (number of segments per chain, energy parameter and segment diameter) are specified for
 329 initialisation. Only an overview of the method is included in this section.

330

331 **Algorithm**

332 The algorithm is summarized in Figure 3. The main steps are:

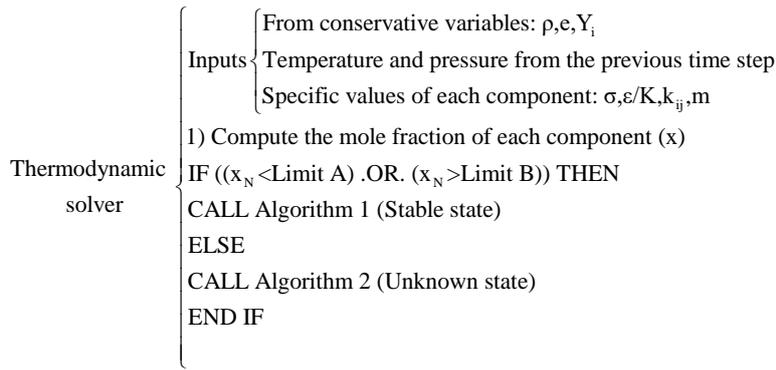
333

334 1) **Filter.** This step is employed to decrease the computational time by reducing VLE
 335 calculations. By checking the molar fractions of the components, it can be determined
 336 whether only one phase exists. Isobaric-adiabatic lines can be computed using the initial
 337 conditions of the case of interest (temperature in the chamber, temperature of the fuel and
 338 pressure in the combustion chamber) to determine the molar fractions at which VLE is not
 339 expected. For example, as we can see in Figure 4, by performing an injection of n-dodecane
 340 at 363K in a combustion chamber at 900K and 11MPa, the nitrogen mole fraction at which
 341 the fuel starts vaporizing is close to 0.15 and there is not liquid phase at nitrogen mole
 342 fractions higher than 0.95. In this case, it would be safe to consider that any mixture with
 343 a nitrogen molar fraction lower than 0.05 (Limit A) and higher than 0.95 (Limit B) will
 344 not be in a VLE state. The reason of choosing a low Limit A is to consider the pressure
 345 variations along the simulation, which have an important effect on the stability of n-
 346 dodecane / N₂ mixtures.

347

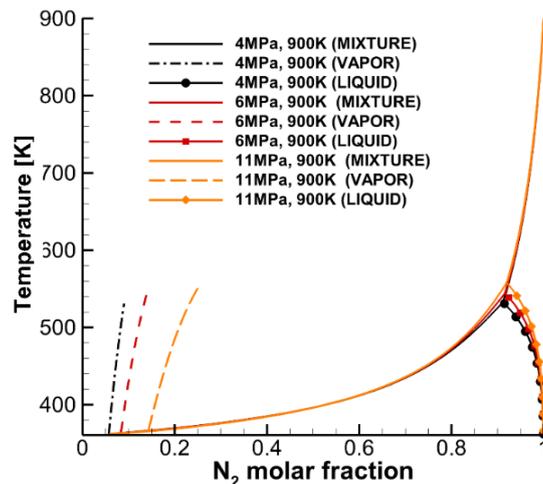
348 2) **Stable state (one phase).** When knowing that the mixture is stable the molecular density
 349 of the mixture can be computed and used as an input to the PC-SAFT model. A Newton
 350 method is employed to compute the temperature that is needed to calculate the value of all
 351 other thermodynamic variables. The temperature dependent function used in the iterative
 352 method is the internal energy. The derivative of the internal energy with respect to the
 353 temperature at constant molecular density can be directly obtained as these are the
 354 independent variables of the PC-SAFT model. See Appendices 2 and 3.

355



356
357 **Figure 3. Schematic representation of the thermodynamic solver**

358



359
360 **Figure 4. Isobaric-adiabatic mixing lines (N₂ + C₁₂H₂₆) at different pressures in the combustion**
361 **chamber**
362

363 3) **Unknown state.** If the state of the mixture is unknown the density cannot be used as an
364 input. The pressure and the temperature are iterated employing a multidimensional Newton
365 method until the density and the internal energy obtained using the PC-SAFT + VLE
366 calculations are the ones obtained from the conservative variables. For each P-T
367 calculation a stability analysis is performed to determine if the mixture is stable. See
368 Appendices 4 and 5.

369
370 a. **Mixture stable:** The PC-SAFT model is solved. The reduced density is
371 iterated until the computed pressure is the input pressure.

372
373 b. **Mixture unstable:** The isothermal-isobaric flash problem (TPn flash) is
374 solved and the properties of the fluid in a VLE state are computed.

375
376 **Stability analysis**

377 A mixture is stable at a specific T and P if the total Gibbs energy is at its global minimum. If
378 an infinitesimal amount (δe) of a new phase of composition \mathbf{w} is formed from a phase of
379 composition \mathbf{z} , the change in the Gibbs energy can be expressed as [44]:

380
$$\delta G = \delta e \sum_{i=1}^C w_i (\mu_i(\mathbf{w}) - \mu_i(\mathbf{z})) \quad (8)$$

381 μ being the chemical potential.

382

383 A necessary condition for the stability of the phase of composition \mathbf{z} is that δG is non-negative
384 for any positive δe for any composition \mathbf{w} . This is known as the tangent plane condition of
385 Gibbs.

$$386 \sum_{i=1}^C w_i (\mu_i(\mathbf{w}) - \mu_i(\mathbf{z})) \geq 0 \quad \forall \quad w_i \geq 0 \quad \text{such that} \quad \sum_{i=1}^C w_i = 1 \quad (9)$$

387

388 The Tangent Plane Distance (TPD) function [45] is employed to determine if a split into two
389 phases decreases the Gibbs energy.

$$390 TPD(\mathbf{w}) = \sum_{i=1}^C w_i (\mu_i(\mathbf{w}) - \mu_i(\mathbf{z})) \quad (10)$$

391 The TPD function can be written in a dimensionless form employing the fugacity coefficient
392 (φ):

$$393 tpd(w_i) = \frac{TPD}{RT} = \sum_{i=1}^C w_i [\ln \varphi_i(\mathbf{w}) + \ln w_i - d_i(\mathbf{z})] \quad (11)$$

394 being

$$395 d_i(\mathbf{z}) = \ln \varphi_i(\mathbf{z}) + \ln z_i$$

396

397 The mixture of composition \mathbf{z} is considered stable if all the TPD local minima are non-negative.

$$398 tpd(\mathbf{w}) \geq 0 \quad \forall \quad w_i \geq 0 \quad \forall \quad w_i \geq 0 \quad \text{such that} \quad \sum_{i=1}^C w_i = 1 \quad (12)$$

399

400 The Successive Substitution Iteration (SSI) algorithm ([16], [46]) (without the Newton
401 method) has been employed to determine if the mixture is stable. See Appendix 6.

402

403 **TPn flash**

404 Once it is known that the mixture is in a VLE state, a multidimensional Newton iteration in T
405 and P is performed until the internal energy and density of the liquid-gas mixture are the ones
406 determined by the conservative variables. An isothermal-isobaric flash problem (known as TPn
407 flash) is performed for each iteration.

408

409 A necessary condition for equilibrium is that the chemical potential for each component is the
410 same in the liquid and vapor phases.

$$411 \mu_i^L = \mu_i^V \quad (13)$$

412 or equivalently using the fugacities:

$$413 f_i^L = f_i^V \quad (14)$$

414

415 Employing the fugacity coefficients, this expression can be written as:

$$416 F_i = \ln \varphi_v(T, p, y) - \ln \varphi_l(T, p, x) + \ln K_i = 0 \quad (15)$$

417 where

$$418 K_i = \frac{x_i}{y_i} = \frac{\varphi_{i,l}}{\varphi_{i,v}} \quad (16)$$

419

420 A successive substitution method is employed to perform equilibrium calculations at specified
421 temperature, pressure and overall composition to determine the liquid and vapor phases that
422 satisfy eq.15 [16], [44]. See Appendix 7.

423

424

425 2.d. VLE interface

426 In [10], molecular dynamic simulations of three n-alkane fuels into nitrogen under
427 various temperatures and pressures were performed to study the injection, evaporation and
428 mixing processes of hydrocarbon fuels into a supercritical environment. The study was focused
429 on understanding the transition from classical two-phase evaporation to one-phase diffusion-
430 controlled mixing. Using as threshold a dimensionless transition time (the time needed to transit
431 from subcritical to supercritical respect to the liquid lifetime) of 0.35, the authors identified two
432 regions on the P-T diagram, see Figure 5. Supercritical dominated: Due to the high critical
433 pressures of TYPE IV mixtures, a VLE state is present at the beginning of the evaporation
434 process. The temperature of the liquid core goes up until the VLE state disappears and only a
435 diffusion-controlled mixing process exists. Subcritical dominated: A clear interface exists
436 between the liquid core and the ambient gases. Nitrogen is not able to diffuse into the liquid
437 core (constant fuel mass fraction close to 1 during evaporation, see Figure 7). There is a gradual
438 decrease of the density of the liquid core as the fuel is heated-up. The evaporation reaches a
439 constant state with a constant liquid core.

440 According to the classification presented by [10], the combustion chamber of a Diesel
441 engine working at medium-high load operation conditions is in the supercritical dominated
442 regime after the compression cycle, see Figure 5. At these ambient conditions, the nitrogen is
443 able to rapidly diffuse into the liquid core indicating that the interface has a Knudsen-number
444 low enough to fall within the fluid mechanic continuum domain [7]. At 20MPa, the molar
445 fraction of nitrogen in the liquid core (before the transition to a diffusion-controlled mixing
446 process) at 0.5ns is almost 20%, see Figure 6. Therefore, the heat-up of the liquid core is
447 dominated by diffusion phenomena. This can be proven by showing how isobaric-adiabatic
448 mixing lines can replicate the heat-up profiles obtained in the molecular simulations of [10].
449 The isobaric-adiabatic lines where computed using eq.17. Figure 7 clearly shows how this
450 hypothesis is not applicable in the subcritical dominated regime where after 5 ns the N₂ molar
451 fraction in the liquid core has a constant value of 2%.

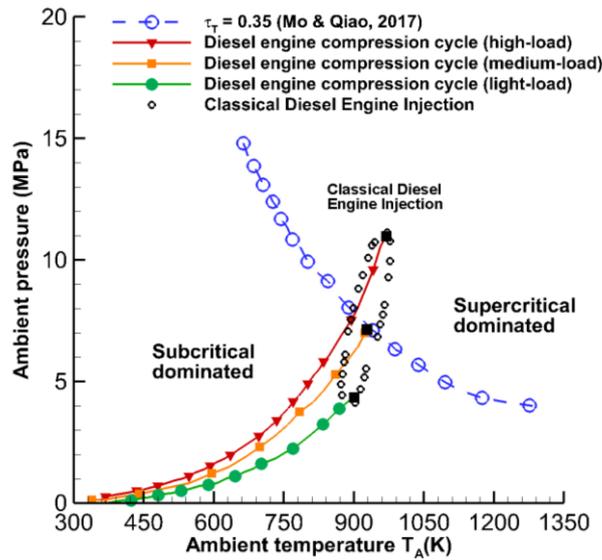
$$452$$
$$453 h_{mixture} = y_{C_{12}H_{26}} h_{C_{12}H_{26}} + y_{N_2} h_{N_2} \quad (17)$$
$$454 p = 20MPa$$

455 being y the mass fraction.

456

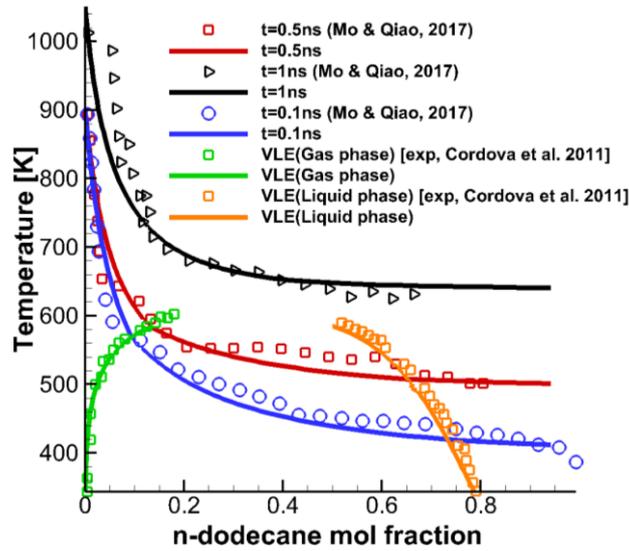
457 The hypothesis employed in this paper is that the vaporization process at high-pressure
458 Diesel fuel injections is located at the subcritical vaporization stage of the supercritical
459 vaporization regime described by [10] without a transition to the diffusion-controlled mixing
460 process. Being the convective forces much more dominant than the diffusion phenomena, N₂-
461 n-dodecane mixing takes place in a time several orders of magnitude lower than the one
462 observed in Figure 6 where only diffusion is present. Thus, the heat-up of the jet describes a
463 single isobaric-adiabatic mixing line instead of multiple adiabatic lines at different times. This
464 can be corroborated observing the results obtained by [15], [16] where the heat-up of the heat
465 follows an isobaric-adiabatic mixing line constant in time solving both, convection and
466 diffusion phenomena in their simulations. A diffuse interface method, which describe an
467 adiabatic heat-up of the jet, must be applied during Diesel engine injection simulations at high-
468 pressure conditions (supercritical dominated regime) to properly characterize how the fuel
469 vaporize.

470



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Figure 5. Diesel engine compression cycles [15] and contours of dimensionless transition time on pressure-temperature diagram of n-dodecane [10]



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476
477
478

Figure 6. Development of gas-liquid interface shown on VLE diagram at 20 MPa [10], VLE experimental data [38] and isobaric-adiabatic mixing lines.

479 3. Results

480 The working fluids employed are the following: (i) n-dodecane, (ii) a mixture of n-octane, n-
481 dodecane and n-hexadecane; (iii) a pseudo-component that replicate the properties of the
482 previous mixture; (iv) V0A Diesel, and (v) a pseudo-component that replicate the properties of
483 the V0A Diesel.

484

485 3.a Shock Tube Problems

486 Shock Tube Problem 1 (One phase, one component)

487 A shock tube problem is used to validate the numerical solution of the hyperbolic operator. The
488 results are compared with an exact solution computed using the methodology described in [47].

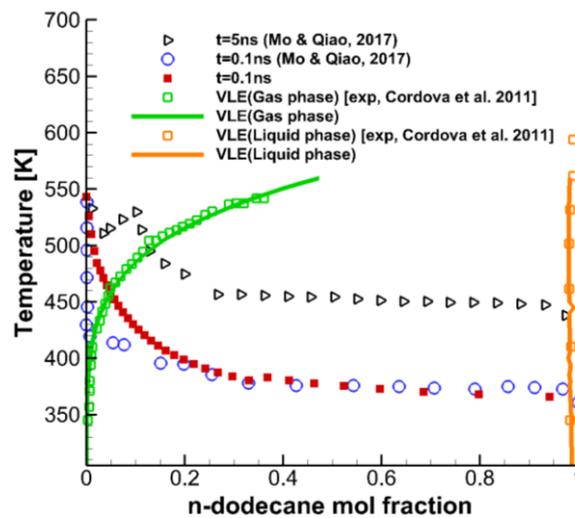
489 N-dodecane is utilized as working fluid; the domain is $x \in [-0.5, 0.5]$ m; 300 equally spaced
 490 cells were employed; wave transmissive boundary conditions are used in the left and right sides;
 491 the simulated time is $5 \cdot 10^{-4}$ s; the initial conditions in the left state are $\rho_L=438\text{kg/m}^3$, $p_L=30\text{MPa}$,
 492 $u_L=0\text{m/s}$; and in the right state are $\rho_R=100\text{kg/m}^3$, $p_R=10\text{MPa}$, $u_R=0\text{m/s}$. Figure 8 shows how the
 493 density, temperature, velocity and pressure results agree with the exact solution.

494

495 **Shock Tube Problem 2 (One phase, multicomponent/pseudo-component)**

496 The working fluids employed are a mixture of n-octane, n-dodecane and n-hexadecane (Table
 497 5) and a pseudo-component that replicate the properties of the mixture (Table 6) [41]. Figure 9
 498 shows a comparison of the results obtained employing the multicomponent mixture and the
 499 results obtained by [41].

500



501

502 **Figure 7. Development of gas–liquid interface shown on VLE diagram at 1 MPa [10], VLE**
 503 **experimental data [38] and isobaric-adiabatic mixing line.**

504

505 The domain is $x \in [-0.5, 0.5]$ m; 800 equally spaced cells were employed; wave transmissive
 506 boundary conditions are used in the left and right sides; the simulated time is $5 \cdot 10^{-4}$ s; the initial
 507 conditions in the left state are $\rho_L=438\text{kg/m}^3$, $t_L=859.5\text{K}$, $u_L=0\text{m/s}$; and in the right state are
 508 $\rho_R=100\text{kg/m}^3$, $t_R=1744\text{K}$, $u_R=0\text{m/s}$. Figure 10 presents the density, temperature, pressure,
 509 velocity, speed of sound and internal energy results. The pseudo-component results are the
 510 same as the multicomponent ones indicating that the methodology developed by [41] can be
 511 used to model complex hydrocarbon mixtures as a pseudo-component in CFD simulations that
 512 present one phase.

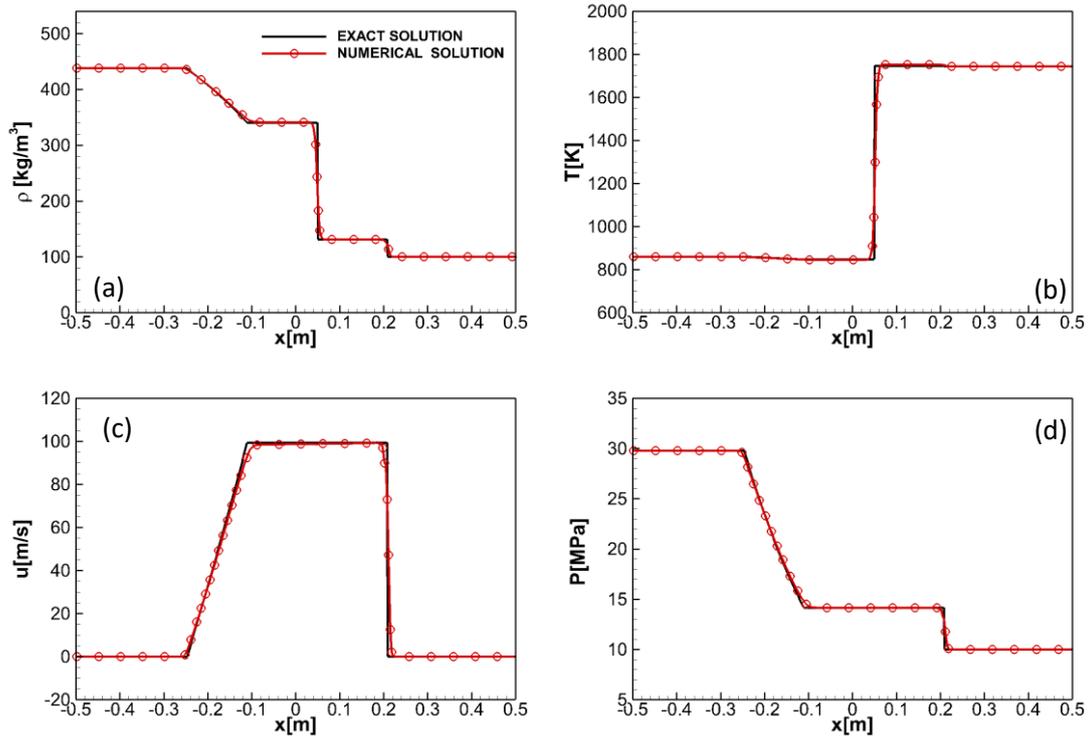
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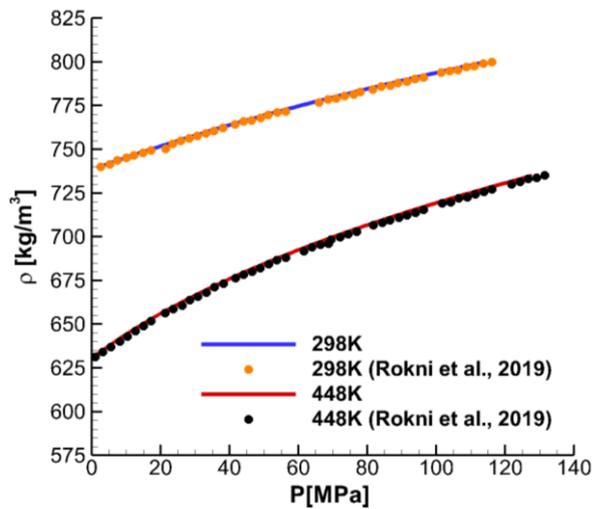
515 **Table 5. Molar composition of hydrocarbon mixture employed in Shock Tube Problem 2 [41]**

Compound	Hydrocarbon mixture (Molar composition)
n-hexadecane	0.232
n-octane	0.460
n-dodecane	0.232

516



517 **Figure 8. Shock Tube Problem ($C_{12}H_{26}$). CFL = 0.5, $u = 0$ m/s, 300 cells, $t=5 \cdot 10^{-4}$ s. Comparisons**
 518 **of (a) density, (b) temperature, (c) velocity and (d) pressure profiles: exact solution and**
 519 **numerical solutions.**
 520



521 **Figure 9. Density predictions for the hydrocarbon mixture presented in Table 5.**
 522
 523
 524
 525

526 **Table 6. PC-SAFT pure component parameters employed to model the pseudo-component**
 527 **employed in Shock Tube Problem 2 [41]**

Compound	m	σ (Å)	ϵ/k (K)
Pseudo-component	7.387	3.400	234.47

528
 529

530 **3.b Advection test cases**

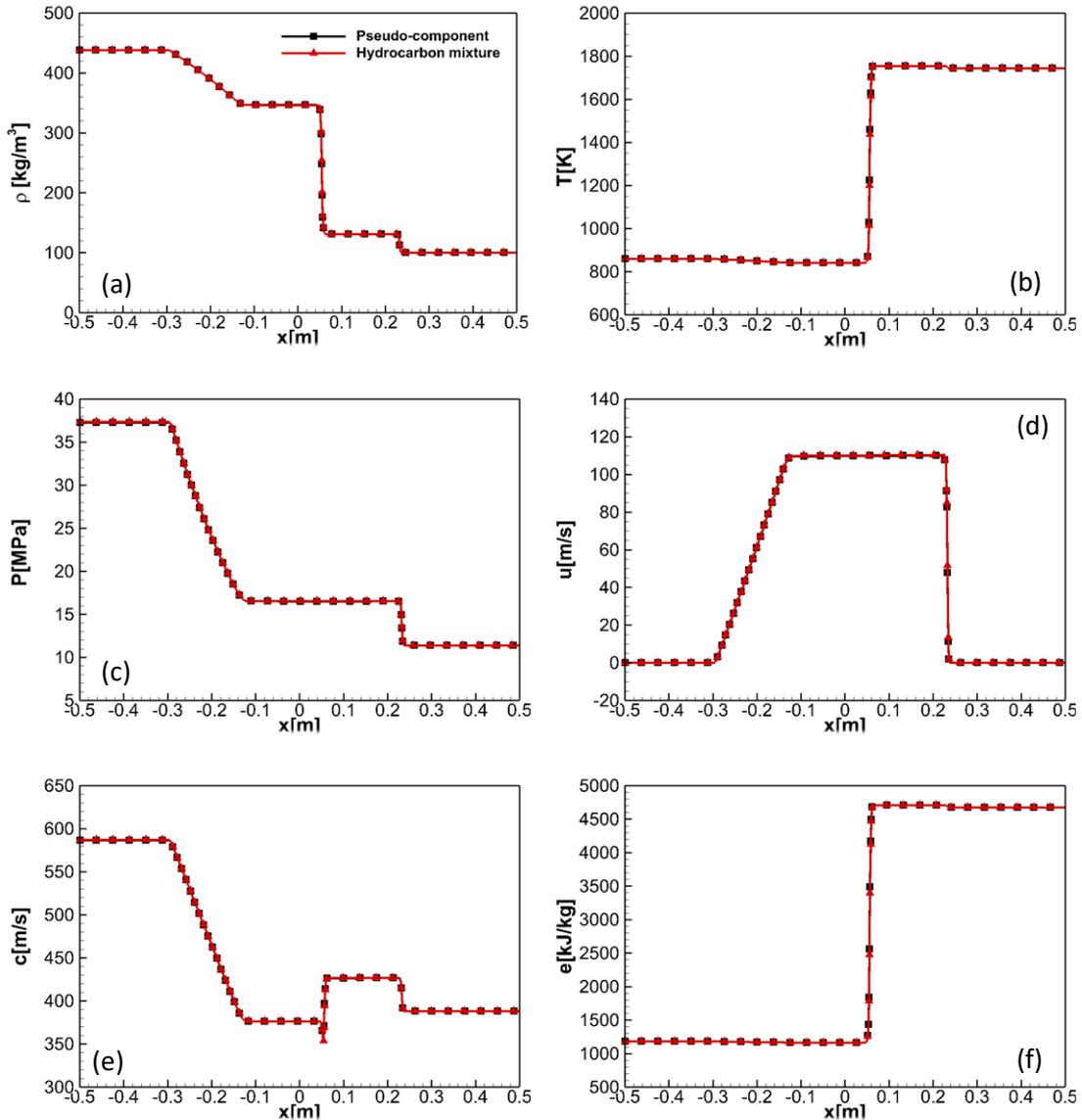
531 The computational domain is $x \in [-10^{-5}, 10^{-5}]$ m; the simulated time is 10^{-6} s; the left initial
 532 conditions are fuel at $p=11$ MPa, $u=10.0$ m/s and $t=362$ K; the right initial conditions are nitrogen
 533 at $p=11$ MPa, $u=10.0$ m/s and $t=972$ K; a uniform grid spacing (100 cells) is applied; CFL is set
 534 to be 0.5; wave transmissive boundary conditions are implemented in the left and right sides of
 535 the computational domain; and a smooth initial interface is applied to reduce the initial start-up
 536 error [48]. When a diffuse interface method is employed, the interfaces are not sharp one-point
 537 jumps but smooth as they are resolved [48]. Thus, a smooth initial profile is a realistic initial
 538 condition. The initial interface was computed employing eq.18 [16] .

539

540
$$Y_{FUEL} = 0.5 - 0.5 \operatorname{erf}\left\{\left(x_1 + 0.25l_{ref}\right)/\left(0.01l_{ref}\right)\right\}$$
 (18)

541

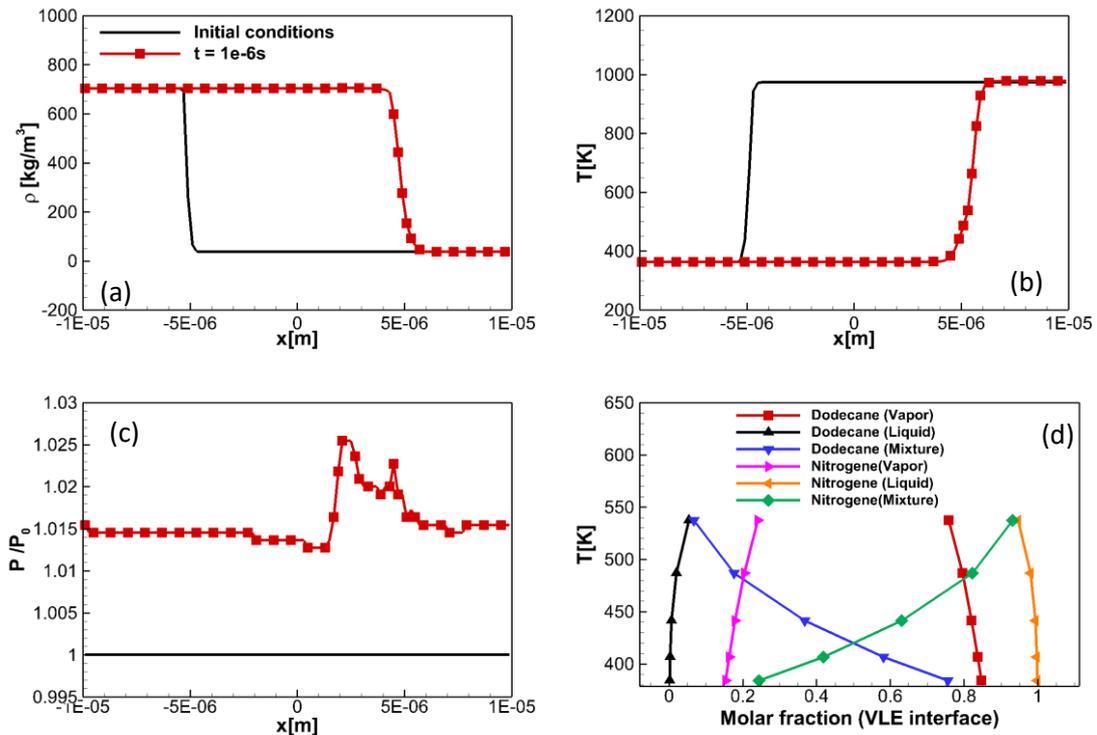
542 The initialization of each cell located in the interface is performed knowing the pressure,
 543 enthalpy of the mixture (eq.17) and the molar fraction of the components.



544 **Figure 10. Shock Tube Problem 2. CFL = 0.5, 800 cells, $t=5 \cdot 10^{-4}$ s. Comparison of the (a) density,**
 545 **(b) temperature, (c) pressure, (d) x-velocity, (e) sonic fluid velocity, (f) internal energy using as**
 546 **working fluids are a mixture of n-octane, n-dodecane and n-hexadecane (Table 5) and a pseudo-**
 547 **component that replicate the properties of the mixture (Table 6) [41].**

548 **Advection test cases 1 (Two phases, n-dodecane/nitrogen)**

549 Figure 11 shows the results of this advection test case where n-dodecane is employed
 550 as fuel. The binary interaction parameter applied is $k_{ij} = 0.1446$. The numerical framework
 551 perfectly captures the large density and temperature gradients present in this multicomponent-
 552 multiphase one-dimensional test case. Small spurious pressure oscillations appear in the
 553 solution. This problem is well known in multicomponent density based codes employing highly
 554 non-linear EoS [16], [19], [33]. Although the small oscillations in the pressure field could be
 555 avoided or reduced employing a QC formulation like the double flux model [32]–[34] or using
 556 a pressure evolution equation [16], [49], [50] instead of the total energy conservation equation
 557 , these schemes presents an error in the energy conservation that produce an unphysical quick
 558 heat-up of the fuel [16], [51]. The combination of VLE + PC-SAFT calculations allows to
 559 properly model: (1) the properties of n-dodecane at high density ranges where cubic models
 560 show large deviations in the sonic fluid velocity (used in density based CFD codes to compute
 561 the hyperbolic fluxes and time step [19]), temperature and internal energy; (2) and a correct
 562 (adiabatic) subcritical evaporation process in the interface (Section 2.d).



563 **Figure 11. Advection Test Case 1 (N₂- C₁₂H₂₆), CFL = 0.5, u = 10 m/s, 100 cells. Results of**
 564 **(a) density, (b) temperature, (c) pressure and (d) VLE interface at 10⁻⁶s.**

566 **Advection test case 2 (Two phases, V0A Diesel/ nitrogen, pseudo - V0A Diesel / nitrogen)**

567 Figure 12 shows the temperature, density, speed of sound and internal energy results of
 568 an advection test case that employs the multicomponent Diesel V0A and the pseudo-Diesel
 569 V0A (Table 7) as fuels. The binary interaction parameter used between the nitrogen and the
 570 Diesel compounds or the pseudo-component is the same one used in the N₂ / n-dodecane
 571 mixture ($k_{ij} = 0.1446$). The pseudo-component presents an error (using as reference the
 572 multicomponent Diesel results) of 1.6% in density, 3.7% in sonic fluid velocity and 5.5% in
 573 internal energy. However, the computational time required to solve the multicomponent V0A
 574 Diesel advection test case is 432% the time consumed by the pseudo-Diesel advection test case.
 575 The different computational requirements will be even bigger in multidimensional cases or

576 simulations where the hydrocarbon mixture presents more components (e.g., V0B, V1 and V2
 577 Diesel surrogates). In the case of the Diesel surrogate V0a, the equilibrium state of five
 578 components must be computed in each cell of the interface, see Figure 13. Using the
 579 methodology of [41], the number of working fluids is limited to 2 (pseudo-Diesel + N₂).

580 Figure 14 shows how the phase boundary from VLE at 11MPa is different if the
 581 multicomponent Diesel V0A or its pseudo-component are employed. The use of a pseudo-
 582 component must not alter how the fuel is heat-up, especially in Diesel injection simulations
 583 where the temperature plays a significant role on determining the ignition time. Figure 15
 584 presents the results in the VLE interface of both working fluids (multicomponent mixture and
 585 pseudo-component).

586

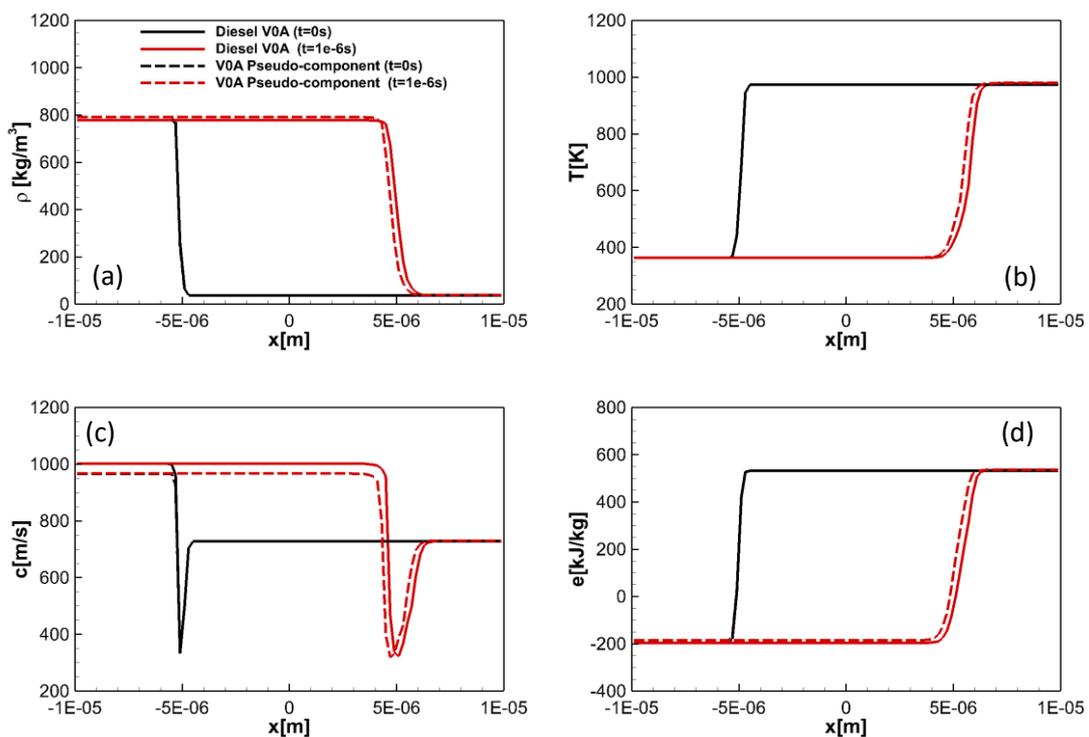
587

588 **Table 7. Pseudo-component PC-SAFT parameters employed to model the pseudo-Diesel V0A**
 589 **using the correlations developed by utilizing the GC parameters of Tihic et al. [42]**

Compound	m	σ (Å)	ε/k (K)
Pseudo-component	5.436	3.908	256.700

590

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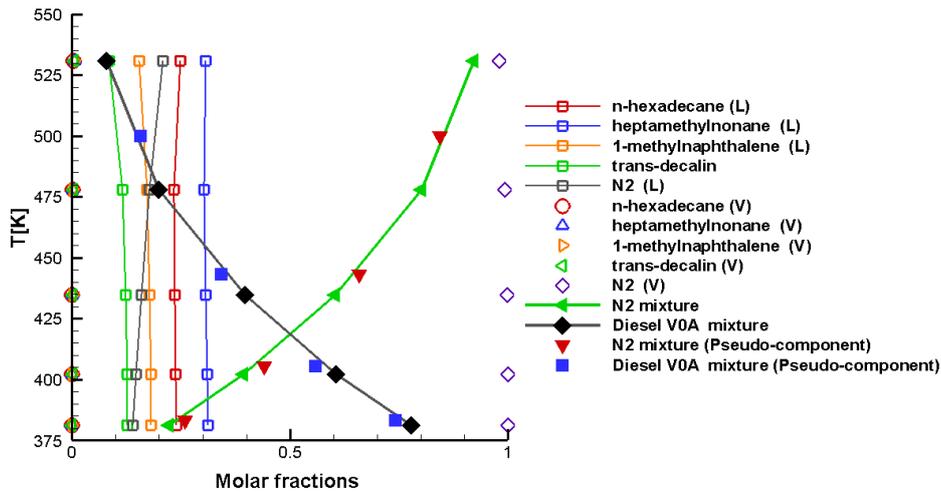
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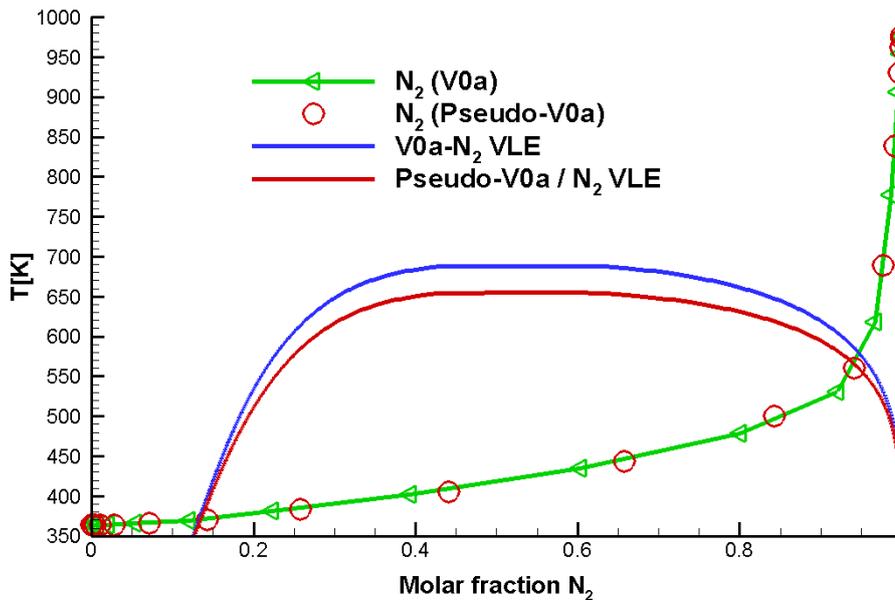
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Figure 12. Advection Test Case 2 (N₂- V0A/ pseudo-Diesel V0A), CFL = 0.5, u = 10 m/s, 100 cells. Results of (a) density, (b) temperature, (c) speed of sound and (d) internal energy results at 10^{-6} s.



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Figure 13. VLE Interface, Advection Test Case 2 (N₂- V0A/ pseudo-Diesel V0A), CFL= 0.5, u = 10 m/s, 100 cells. Results of VLE interface at 10⁻⁶s.



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Figure 14. Advection Test Case 2 (N₂- V0A/ pseudo-Diesel V0A). Results of VLE interface at 10⁻⁶s and phase boundaries from VLE at 11MPa.

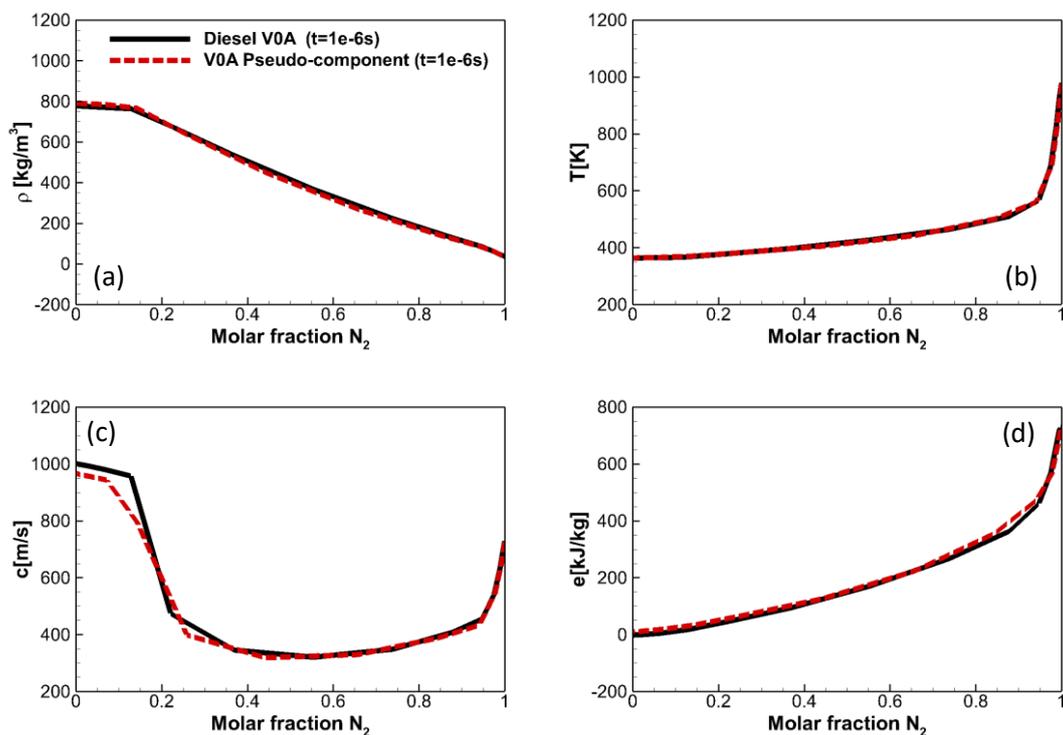
602 3.c Two-dimensional cases

603 Planar two-dimensional injections of n-dodecane and a Diesel pseudo-component are
604 presented to demonstrate the multidimensional capability of the numerical framework.

605 N-dodecane jet

606 A structured mesh is applied with a uniform cell distribution; the domain used is 12mm
607 × 6mm; 1,216,800 cells are employed; the parabolic sub-step is included into these simulations
608 without sub-grid scale modelling for turbulence or heat/species diffusion; the CFL number is
609 set at 0.5; the fifth-order WENO discretization scheme presented in [18] is used; transmissive
610 boundary conditions are applied at the top, bottom and right boundaries while a wall condition
611 is employed at the left boundary; a flat velocity profile is imposed at the inlet; the velocity of

612 the jet is 600 m/s; the diameter of the exit nozzle is 0.1mm; the case is initialized using a
613 pressure in the chamber of 11 MPa; the temperature of the nitrogen is 973 K; and the
614 temperature of the injected fuel is 363K. The binary interaction parameter applied is $k_{ij} =$
615 0.1446. The loops where the hyperbolic fluxes, parabolic fluxes, update of conservative
616 variables and thermodynamic solver are solved (see Figure 2) were paralleled employing
617 OpenMP (24 physical cores were employed). Some instabilities were observed in the
618 initialization as [16] reported. To solve this problem, a ramp is used to accelerate the fuel to
619 600m/s. The jet is quickly heated-up from a compressed liquid state to gas and finally, to a
620 supercritical state describing an isobaric-adiabatic mixing line, see Figure 16. Figure 17 shows
621 how the Kelvin Helmholtz instability and ligament-shaped structures are developed in the shear
622 layer.
623

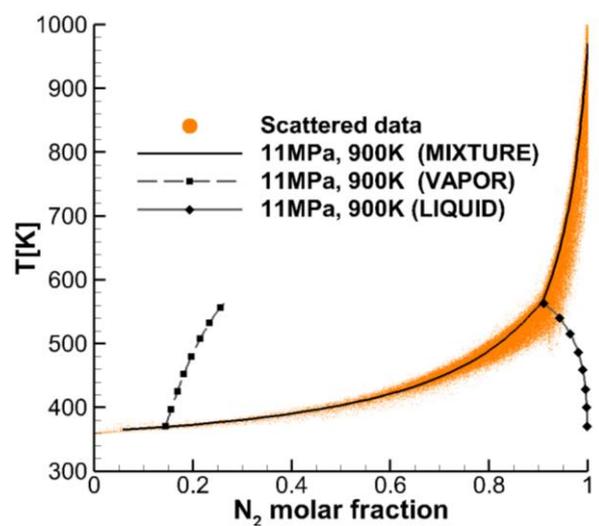


624 **Figure 15. Advection Test Case 2 (N_2 - V0A/ pseudo-Diesel V0A), CFL = 0.5, $u = 10$ m/s,**
625 **100 cells. Results of (a) density, (b) temperature, (c) speed of sound and (d) internal energy**
626 **results at 10^{-6} s.**
627

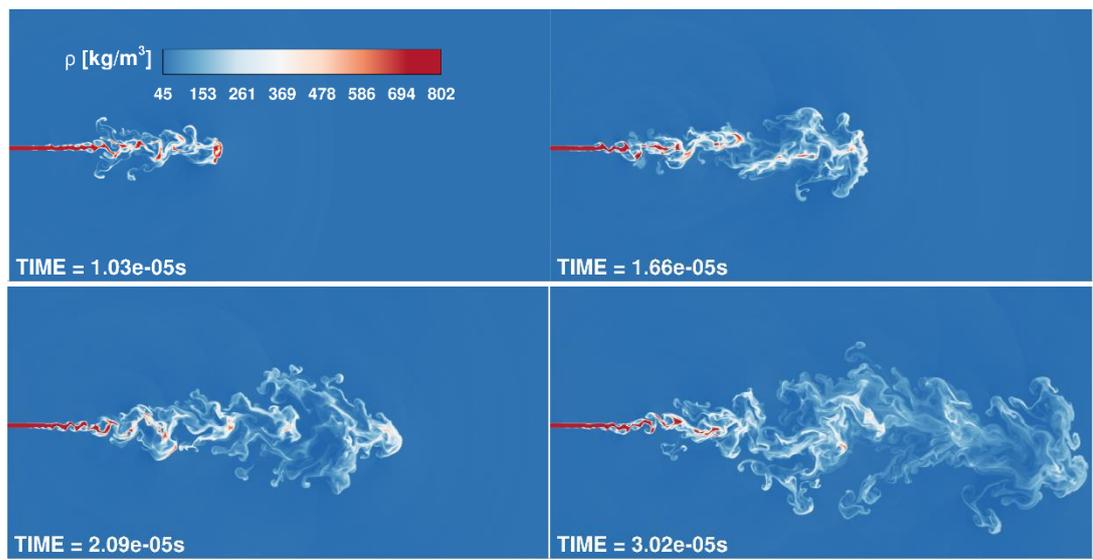
628 Diesel jet

629 The initial conditions and set-up of the simulation is the same as the n-dodecane jet.
630 The binary interaction parameter applied is $k_{ij} = 0.1446$. Figure 18 shows the density,
631 temperature and pressure at 3.19×10^{-5} s. Spurious pressure oscillations are not present in the
632 pressure field despite the multicomponent nature of the simulations and large density gradients
633 solved. The simulation present supersonic, transonic and subsonic regions due to the low values
634 of the speed of sound present in the cells in a VLE state and the high jet velocity, see Figure
635 18. Such a variety of Mach numbers in a simulation can introduce important stability issues
636 However, stability problems were not observed. The computational time required to solve at
637 3.3×10^{-5} s was 91.7 hours. Most time is invested on solving the multidimensional Newton
638 method of the cells that are in a VLE state. At these conditions, the derivatives of the Jacobian
639 matrix are calculated numerically (Appendix 5). However, the developed methodology is fast

640 enough to perform simulations at affordable time scales. It should also be considered that the
 641 results are equivalent to a multicomponent injection of a Diesel surrogate of 4 components that
 642 vaporize when mixed with hot nitrogen.
 643



644 **Figure 16. Scattered data of composition and temperature of the planar n-dodecane jet,**
 645 **dodecane-nitrogen phase boundary from VLE at 11MPa and isobaric-adiabatic mixing line.**
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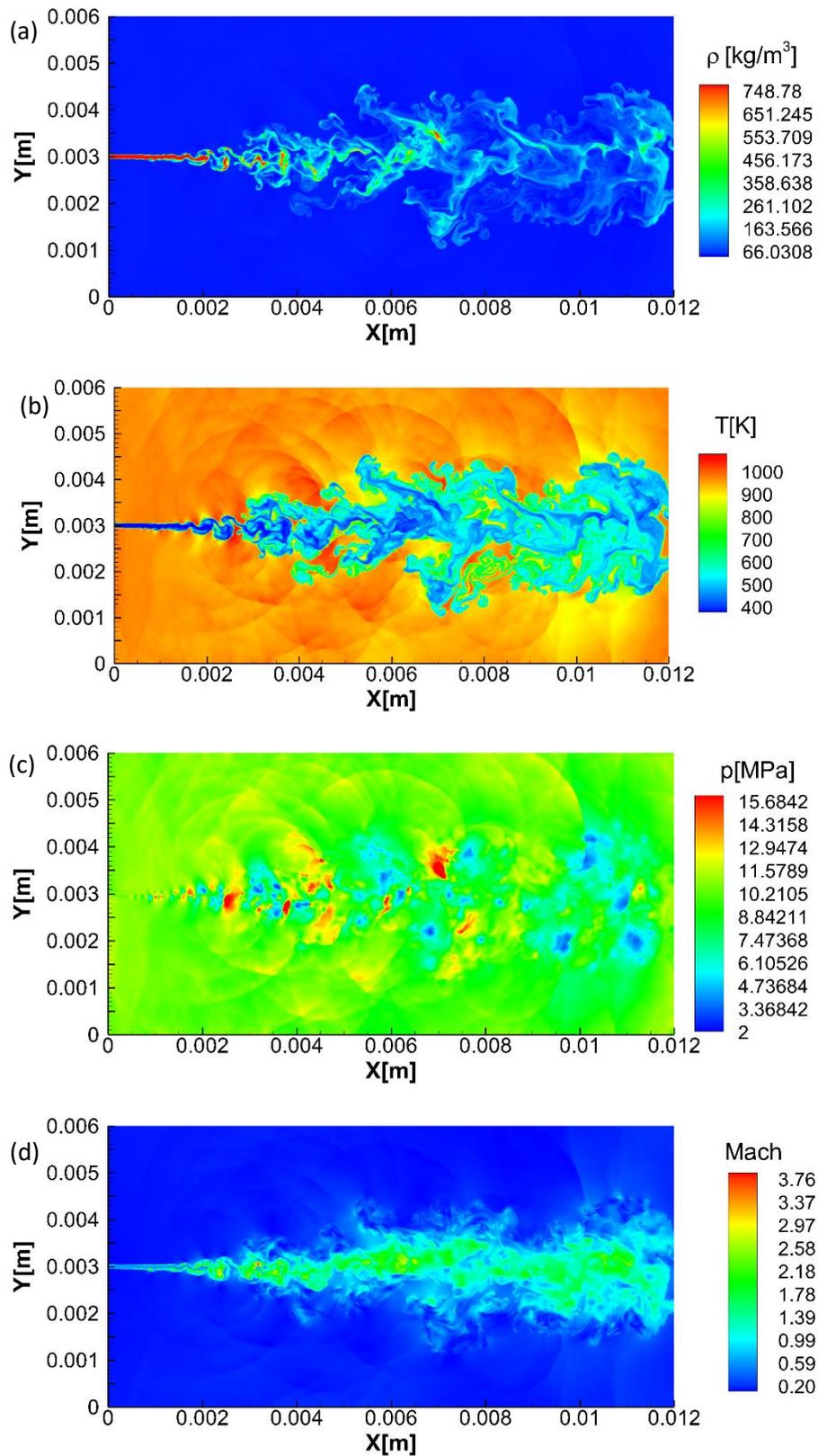
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 651 **Figure 17. Density results of n-dodecane planar jet.**

652 **4. Conclusions**

653 This paper presents a numerical framework that combines PC-SAFT and VLE calculations in
 654 a density-based, fully conservative solver of the Navier-Stokes and energy conservation
 655 equations to simulate fuel-air mixing at high-pressure conditions. This molecular-based EoS
 656 requires three empirically determined but well-known parameters to model the properties of a
 657 specific component, and thus, there is no need for extensive model calibration, as is typically
 658 the case when the NIST (REFPROP) library is utilised. PC-SAFT can flexibly handle the
 659 thermodynamic properties of multi-component mixtures for which the NIST (REFPROP)
 660 library supports only limited component combinations. Modelling multicomponent Diesel

661 surrogates, the PC-SAFT EoS shows the highest degree of agreement with experimental values
662 in comparison with the results obtained using the model developed at NIST. Moreover, a purely
663 predictive method that employs the PC-SAFT EoS for developing pseudo-components has been
664 completed and validated to be used in CFD simulations. Complex hydrocarbon mixtures can
665 be modelled as a single pseudo-component knowing its MW and HN/CN ratio. By employing
666 pseudo-components, the simulation time is independent of the number of compounds present
667 in the fuel and thus, allowing real fuel compositions to be utilised in CFD simulations.
668 Advection test cases and shock tube problems were performed to validate the numerical
669 framework using analytical and exact solutions. The two-dimensional simulations performed
670 (subcritical injections of n-dodecane and Diesel into nitrogen) demonstrate the
671 multidimensional, multispecies and multiphase capability of the algorithm and its high stability
672 in simulations where all sonic regimes are present.

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687 **Figure 18.** CFL = 0.5, 1,216,800 cells. Results of the simulation of the V0A Diesel pseudo-
 688 component jet at $t = 3.19 \times 10^{-5}$ s: (a) density, (b) temperature, (c) pressure and (d) Mach number.
 689
 690

691 **Appendix 1 (CFD CODE)**

692 The Navier-Stokes equations for a non-reacting multi-component mixture containing N species
 693 in a x-y 2D Cartesian system are given by:

694
 695
$$\frac{\partial \mathbf{U}}{\partial t} + \frac{\partial \mathbf{F}}{\partial x} + \frac{\partial \mathbf{G}}{\partial y} = \frac{\partial \mathbf{F}_v}{\partial x} + \frac{\partial \mathbf{G}_v}{\partial y} \quad (\text{A.1})$$

696
 697 The vectors of A.1 are:

698
$$\mathbf{U} = \begin{bmatrix} \rho Y_1 \\ \vdots \\ \rho Y_N \\ \rho u \\ \rho v \\ \rho E \end{bmatrix}, \mathbf{F} = \begin{bmatrix} \rho u Y_1 \\ \vdots \\ \rho u Y_N \\ \rho u^2 + p \\ \rho uv \\ (\rho E + p)u \end{bmatrix}, \mathbf{G} = \begin{bmatrix} \rho v Y_1 \\ \vdots \\ \rho v Y_N \\ \rho vu \\ \rho v^2 + p \\ (\rho E + p)v \end{bmatrix}, \mathbf{F}_v = \begin{bmatrix} J_{x,1} \\ \vdots \\ J_{x,N} \\ \sigma_{xx} \\ \sigma_{xy} \\ u\sigma_{xx} + v\sigma_{xy} - q_x \end{bmatrix}, \quad (\text{A.2})$$

699
$$\mathbf{G}_v = \begin{bmatrix} J_{y,1} \\ \vdots \\ J_{y,N} \\ \sigma_{yx} \\ \sigma_{yy} \\ u\sigma_{yx} + v\sigma_{yy} - q_y \end{bmatrix}$$

700 where ρ is the fluid density, u and v are the velocity components, p is the pressure, E is the total
 701 energy, J_i is the mass diffusion flux of species i , σ is the deviatoric stress tensor and q is the
 702 diffusion heat flux vector.

703
 704 **Hyperbolic sub-step**

705 The multicomponent HLLC (Harten-Lax-van Leer-Contact) solver is applied to solve the
 706 Riemann problem [36]. The fluxes are computed as:

707
 708
$$\mathbf{F}^{\text{HLLC}} = \begin{cases} \mathbf{F}_L & \text{if } 0 \leq S_L, \\ \mathbf{F}_{*L} = \mathbf{F}_L + S_L(\mathbf{U}_{*L} - \mathbf{U}_L) & \text{if } S_L \leq 0 \leq S_*, \\ \mathbf{F}_{*R} = \mathbf{F}_R + S_R(\mathbf{U}_{*R} - \mathbf{U}_R) & \text{if } S_* \leq 0 \leq S_{*R}, \\ \mathbf{F}_R & \text{if } 0 \geq S_{*R}, \end{cases} \quad (\text{A.3})$$

709 where \mathbf{U}_* are the star states [36].

710
 711 The speed in the middle wave is:

712
$$S_* = \frac{p_R - p_L + \rho_L u_L (S_L - u_L) - \rho_R u_R (S_R - u_R)}{\rho_L (S_L - u_L) - \rho_R (S_R - u_R)} \quad (\text{A.4})$$

713
 714 The left and right wave speeds are computed as:

715
$$S_L = \min(u_L - a_L, u_R - a_R),$$

 716
$$S_R = \max(u_L + a_L, u_R + a_R) \quad (\text{A.5})$$

717 The conservative variables, pressure and speed of sound values needed to solve the
 718 Riemann problem are interpolated at the cell faces from cell centers using the fifth order
 719 reconstruction scheme described in [18]. This technique decreases the computational time and
 720 smooths-out the spurious pressure oscillations associated with fully conservative (FC) schemes
 721 employed along with real-fluid EoS.

722

723 The temporal integration is carried out using a second-order Runge–Kutta (RK2)
 724 scheme (A.6) applying the filter presented in [18].

725

$$726 \quad \mathbf{U}^{(1)} = \mathbf{U}^n + \Delta t H_{xy}(\mathbf{U}^n),$$

$$727 \quad \mathbf{U}^{n+1} = \frac{1}{2} \mathbf{U}^n + \frac{1}{2} [\mathbf{U}^{(1)} + \Delta t H_{xy}(\mathbf{U}^{(1)})] \quad (\text{A.6})$$

728

729 Parabolic sub-step

730 The model developed by [37] is used to calculate the dynamic viscosity and the thermal
 731 conductivity. The viscous stress tensor is calculated as:

732

$$733 \quad \sigma_{xx} = 2\mu_v \frac{\partial u}{\partial x} - \frac{2}{3} \mu_v \left(\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right)$$

$$734 \quad \sigma_{yy} = 2\mu_v \frac{\partial v}{\partial y} - \frac{2}{3} \mu_v \left(\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} \right)$$

$$735 \quad \sigma_{xy} = \sigma_{yx} = \mu_v \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \quad (\text{A.7})$$

736

737 where μ_v is the viscosity.

738

739 The species mass diffusion flux of species i is calculated employing Fick's law:

$$740 \quad J_i = \rho D_i \nabla Y_i - Y_i \sum_{j=1}^{Nc} \rho D_j \nabla Y_j \quad (\text{A.8})$$

741 where

$$742 \quad D_i = \frac{(1-z_i)}{\sum_{j \neq i}^{Nc} \frac{z_j}{D_j}} \quad (\text{A.9})$$

743 being D is the diffusion coefficient for the diffusion of the component i in the rest of the mixture
 744 [52].

745

746 The heat flux vector is calculated as:

$$747 \quad \mathbf{q} = -\lambda \nabla T - \sum_i^N h_i J_i \quad (\text{A.10})$$

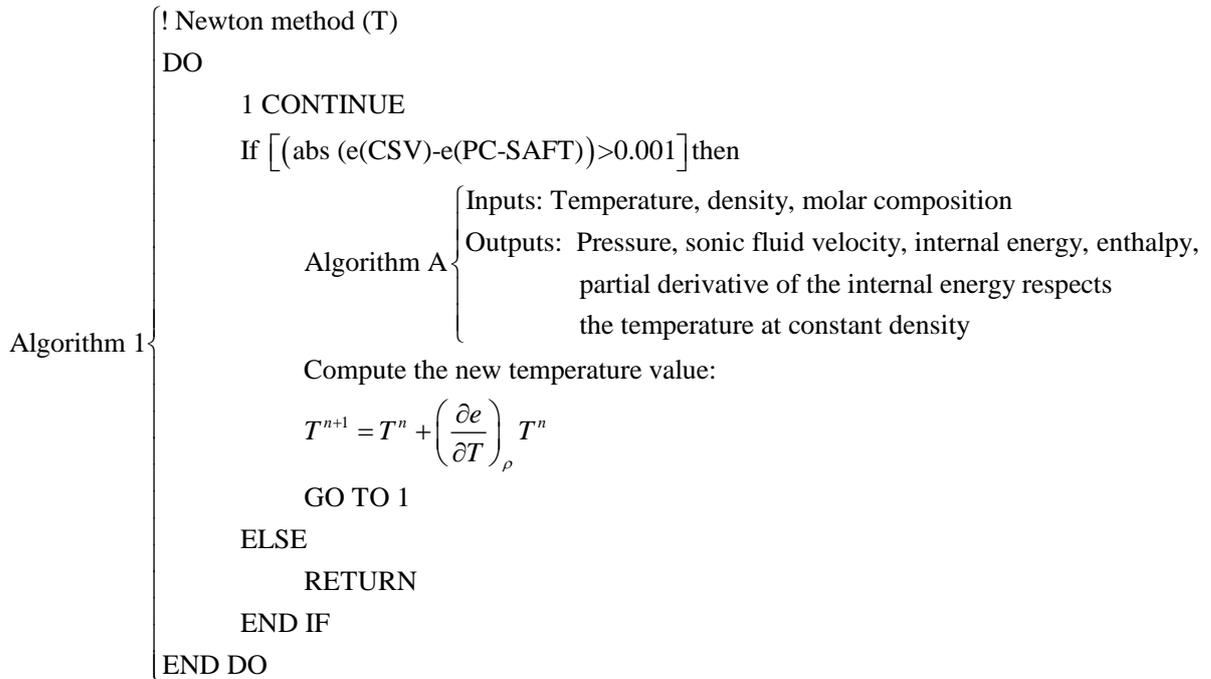
748 where λ is the thermal conductivity and h is the enthalpy.

749

750

751 **Appendix 2 (Thermodynamic solver for stable mixtures)**

752 The molecular density is computed using the density of the mixture. Once the molecular density
 753 is known a Newton method is employed to compute the temperature that is needed to calculate
 754 the value of all other thermodynamic variables. The temperature dependent function used in the
 755 iterative method is the internal energy. Initially a temperature value is assumed (for example
 756 the value of the temperature from the previous time RK sub-step or from the previous time step)
 757 to initialize the iteration process. In most cells, this value is close to the solution.
 758



759
 760

Figure A1. Schematic representation of the Algorithm 1

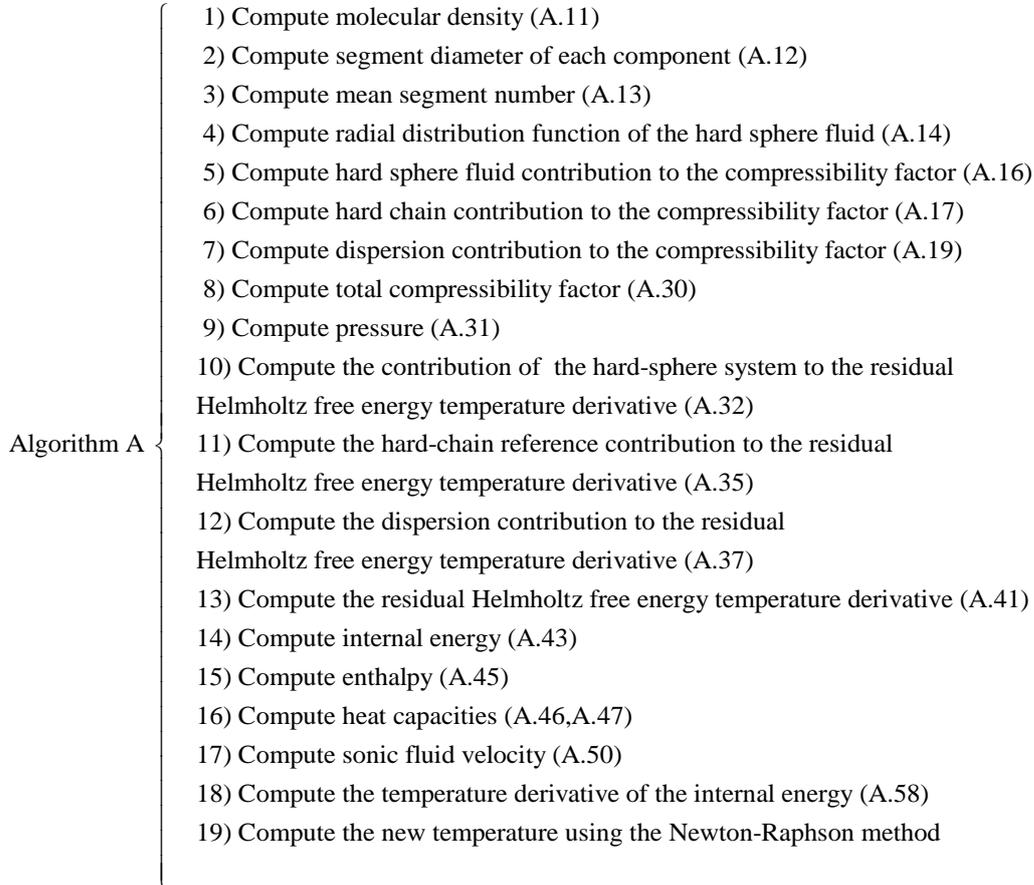
761
 762
 763

764 **Appendix 3 (Algorithm A)**

765 **Inputs:** Temperature, density, molar composition.

766 **Output:** Pressure, sonic fluid velocity, internal energy, enthalpy, partial derivative of the
 767 internal energy respects the temperature at constant density.

768



769

770

Figure A2. Schematic representation of the Algorithm A

771 **Steps**

772 **1) Compute molecular density**

773
$$\rho_m = \rho(kg/m^3) * 10^{-30} * N_A / M_M * 1000 \quad (A.11)$$

774 where N_A is the Avogadro number and M_M is the molecular weight of the mixture.

775

776 **2) Compute temperature-dependent segment diameter d of component i [53]**

777
$$d_i = \sigma_{di} \left[1 - 0.12 \exp\left(-3 \frac{\varepsilon_i}{kT}\right) \right] \quad (A.12)$$

778 where k is the Boltzmann constant, T is the temperature, ε_i is the depth of pair potential
 779 of the component and σ_{di} is the segment diameter.

780

781 **3) Compute mean segment number [53]**

782
$$\bar{m} = \sum_i^{nc} x_i m_i \quad (A.13)$$

783 where m_i is the number of segments per chain of the component i and x_i is the mole
 784 fraction of component i .

785

786 **4) Compute radial distribution function of the hard-sphere fluid [53]**

$$g_{ij}^{hs} = \frac{1}{(1-\zeta_3)} + \left(\frac{d_i d_j}{d_i + d_j}\right) \frac{3\zeta_2}{(1-\zeta_3)^2} + \left(\frac{d_i d_j}{d_i + d_j}\right)^2 \frac{3\zeta_2^2}{(1-\zeta_3)^3} \quad (\text{A.14})$$

where

$$\zeta_n = \frac{\pi}{6} \rho_m \sum_i x_i m_i d_i^n \quad n \in \{0,1,2,3\} \quad (\text{A.15})$$

5) **Compute contribution of the hard sphere to the compressibility factor** [53]

$$Z^{hs} = \frac{\zeta_3}{(1-\zeta_3)} + \frac{3\zeta_1\zeta_2}{\zeta_0(1-\zeta_3)^2} + \frac{3\zeta_2^3 - \zeta_3\zeta_2^3}{\zeta_0(1-\zeta_3)^3} \quad (\text{A.16})$$

6) **Compute hard-chain contribution to the compressibility factor** [53]

$$Z^{hc} = \bar{m}Z^{hs} - \sum_i x_i (m_i - 1) (g_{ii}^{hs})^{-1} \rho_m \frac{\partial g_{ii}^{hs}}{\partial \rho_m} \quad (\text{A.17})$$

$$\rho \frac{\partial g_{ij}^{hs}}{\partial \rho} = \frac{\zeta_3}{(1-\zeta_3)^2} + \left(\frac{d_i d_j}{d_i + d_j}\right) \left(\frac{3\zeta_2}{(1-\zeta_3)^2} + \frac{6\zeta_2\zeta_3}{(1-\zeta_3)^3}\right) + \left(\frac{d_i d_j}{d_i + d_j}\right)^2 \left(\frac{4\zeta_2^2}{(1-\zeta_3)^3} + \frac{6\zeta_2^2\zeta_3}{(1-\zeta_3)^4}\right) \quad (\text{A.18})$$

7) **Compute dispersion contribution to the compressibility factor** [53]

$$Z^{disp} = -2\pi\rho_m \frac{\partial(\eta I_1)}{\partial \eta} \overline{m^2 \varepsilon \sigma_d^3} - \pi\rho_m \bar{m} \left[C_1 \frac{\partial(\eta I_2)}{\partial \eta} + C_2 \eta I_2 \right] \overline{m^2 \varepsilon^2 \sigma_d^3} \quad (\text{A.19})$$

C_1 and C_2 are defined as:

$$C_1 = \left(1 + Z^{hc} + \rho \frac{\partial Z^{hc}}{\partial \rho} \right)^{-1} = \left(1 + \bar{m} \frac{8\eta - 8\eta^2}{(1-\eta)^4} + (1 - \bar{m}) \frac{20\eta - 27\eta^2 + 12\eta^3 - 2\eta^4}{[(1-\eta)(2-\eta)]^2} \right)^{-1} \quad (\text{A.20})$$

$$C_2 = \frac{\partial C_1}{\partial \eta} = -C_1^2 \left(\bar{m} \frac{-4\eta^2 + 20\eta + 8}{(1-\eta)^5} + (1 - \bar{m}) \frac{2\eta^3 + 12\eta^2 - 48\eta + 40}{[(1-\eta)(2-\eta)]^3} \right) \quad (\text{A.21})$$

The terms $\overline{m^2 \varepsilon \sigma_d^3}$ and $\overline{m^2 \varepsilon^2 \sigma_d^3}$ are defined as:

$$\overline{m^2 \varepsilon \sigma_d^3} = \sum_i^{nc} \sum_j^{nc} x_i x_j m_i m_j \left(\frac{\varepsilon_{ij}}{kT}\right) \sigma_{d,ij}^3 \quad (\text{A.22})$$

$$\overline{m^2 \varepsilon^2 \sigma_d^3} = \sum_i^{nc} \sum_j^{nc} x_i x_j m_i m_j \left(\frac{\varepsilon_{ij}}{kT}\right)^2 \sigma_{d,ij}^3 \quad (\text{A.23})$$

The mixture parameters σ_{ij} and ε_{ij} , which are defined for every pair of unlike segments, are modelled using a Berthelot-Lorentz combining rule.

$$\sigma_{ij} = \frac{1}{2} (\sigma_i + \sigma_j) \quad (\text{A.24})$$

$$\varepsilon_{ij} = \sqrt{\varepsilon_i \varepsilon_j} (1 - k_{ij}) \quad (\text{A.25})$$

$\frac{\partial(\eta I_1)}{\partial \eta}$ and $\frac{\partial(\eta I_2)}{\partial \eta}$ are expressed as:

$$\frac{\partial(\eta I_1)}{\partial \eta} = \sum_{j=0}^6 a_j (\bar{m})(j+1)\eta^j \quad (\text{A.26})$$

$$\frac{\partial(\eta I_2)}{\partial \eta} = \sum_{j=0}^6 b_j (\bar{m})(j+1)\eta^j \quad (\text{A.27})$$

The coefficients a and b depend on the chain length:

$$a_i(\bar{m}) = a_{0i} + \frac{\bar{m}-1}{\bar{m}} a_{1i} + \frac{\bar{m}-1}{\bar{m}} \frac{\bar{m}-2}{\bar{m}} a_{2i} \quad (\text{A.28})$$

823 $b_i(\bar{m}) = b_{0i} + \frac{\bar{m}-1}{\bar{m}} b_{1i} + \frac{\bar{m}-1}{\bar{m}} \frac{\bar{m}-2}{\bar{m}} b_{2i}$ (A.29)

824 $a_{0i}, a_{1i}, a_{2i}, b_{0i}, b_{1i}, b_{2i}$ are constants [53].

825

826 **8) Compute compressibility factor [53]**

827 $Z = 1 + Z^{hc} + Z^{disp}$ (A.30)

828

829 **9) Compute pressure [53]**

830 $P = Zk_B T \rho_m (10^{10})^3$ (A.31)

831

832 **10) Compute temperature derivative of the Helmholtz free energy residual**
 833 **contribution of the hard-sphere system [53]**

834

835
$$\left(\frac{\partial \bar{a}^{hs}}{\partial T}\right)_{\rho, x_i} = \frac{1}{\zeta_0} \left[\frac{3(\zeta_{1,T}\zeta_2 + \zeta_1\zeta_{2,T})}{(1-\zeta_3)} + \frac{3\zeta_1\zeta_2\zeta_{3,T}}{(1-\zeta_3)^2} + \frac{3\zeta_2^2\zeta_{2,T}}{\zeta_3(1-\zeta_3)^2} + \frac{\zeta_2^3\zeta_{3,T}(3\zeta_3-1)}{\zeta_3^2(1-\zeta_3)^3} + \right.$$

$$\left. \left(\frac{3\zeta_2^2\zeta_{2,T}\zeta_3 - 2\zeta_2^3\zeta_{3,T}}{\zeta_3^3}\right) \ln(1-\zeta_3) + \left(\zeta_0 - \frac{\zeta_2^3}{\zeta_3^2}\right) \frac{\zeta_{3,T}}{(1-\zeta_3)} \right]$$
 (A.32)

836

837 with abbreviations for two temperature derivatives:

838 $\zeta_{n,T} = \frac{\partial \zeta_n}{\partial T} = \frac{\pi}{6} \rho \sum_i x_i m_i n d_{i,T} (d_i)^{n-1} \quad n \in \{0,1,2,3\}$ (A.33)

839 $d_{i,T} = \frac{\partial d_i}{\partial T} = \sigma_i \left(3 \frac{\varepsilon_i}{kT^2}\right) \left[-0.12 \exp\left(-3 \frac{\varepsilon_i}{kT}\right)\right]$ (A.34)

840

841 **11) Compute temperature derivative of the Helmholtz free energy hard-chain**
 842 **reference contribution [53]**

843

844 $\left(\frac{\partial \bar{a}^{hc}}{\partial T}\right)_{\rho, x_i} = \bar{m} \left(\frac{\partial \bar{a}^{hs}}{\partial T}\right)_{\rho, x_i} - \sum_i x_i (m_i - 1) (g_{ii}^{hs})^{-1} \left(\frac{\partial g_{ii}^{hs}}{\partial T}\right)_{\rho, x_i}$ (A.35)

845

846 The temperature derivative of the radial pair distribution function is:

847
$$\frac{\partial g_{ii}^{hs}}{\partial T} = \frac{\zeta_{3,T}}{(1-\zeta_3)^2} + \left(\frac{1}{2} d_{i,T}\right) \frac{\zeta_2}{(1-\zeta_3)^2} + \left(\frac{1}{2} d_i\right) \left(\frac{3\zeta_{2,T}}{(1-\zeta_3)^2} + \frac{6\zeta_2\zeta_{3,T}}{(1-\zeta_3)^3}\right) +$$

$$\left(\frac{1}{2} d_i d_{i,T}\right) \frac{2\zeta_2^2}{(1-\zeta_3)^3} + \left(\frac{1}{2} d_i\right)^2 \left(\frac{4\zeta_2\zeta_{2,T}}{(1-\zeta_3)^3} + \frac{6\zeta_2^2\zeta_{3,T}}{(1-\zeta_3)^4}\right)$$
 (A.36)

849

850 **12) Compute temperature derivative of the Helmholtz free energy dispersive**
 851 **attraction [53]**

852

853
$$\left(\frac{\partial \bar{a}^{disp}}{\partial T}\right)_{\rho, x_i} = -2\pi\rho \left(\frac{\partial I_1}{\partial T} - \frac{I_1}{T}\right) \overline{m^2 \varepsilon \sigma_d^3} - \pi\rho\bar{m}$$

$$\left[\frac{\partial C_1}{\partial T} I_2 + C_1 \frac{\partial I_2}{\partial T} - 2C_1 \frac{I_2}{T}\right] \overline{m^2 \varepsilon^2 \sigma_d^3}$$
 (A.37)

855 with

856 $\frac{\partial I_1}{\partial T} = \sum_{i=0}^6 a_i(\bar{m}) i \zeta_{3,T} \eta^{i-1}$ (A.38)

857 $\frac{\partial I_2}{\partial T} = \sum_{i=0}^6 b_i(\bar{m}) i \zeta_{3,T} \eta^{i-1}$ (A.39)

$$858 \quad \frac{\partial C_1}{\partial T} = \zeta_{3,T} C_2 \quad (A.40)$$

859

860 **13) Compute temperature derivative of the Helmholtz free energy [53]**

861

$$862 \quad \left(\frac{\partial \bar{a}^{res}}{\partial T} \right)_{\rho, x_i} = \left(\frac{\partial \bar{a}^{hc}}{\partial T} \right)_{\rho, x_i} + \left(\frac{\partial \bar{a}^{disp}}{\partial T} \right)_{\rho, x_i} \quad (A.41)$$

863

864 **14) Compute the internal energy [54]**

865 The internal energy is estimated as the sum of the ideal internal energy and the residual
866 internal energy [54].

$$867 \quad \frac{e^{res}}{RT} = -T \left(\frac{\partial \bar{a}^{res}}{\partial T} \right)_{\rho, x_i} \quad (A.42)$$

$$868 \quad e = e^{res} + e^{id} \quad (A.43)$$

869

870 **15) Compute enthalpy [53]:**

871 It is computed as the sum of the ideal contribution (obtained by integrating the ideal
872 heat capacity at constant pressure with respect to the temperature) and the residual
873 enthalpy [53].

$$874 \quad \frac{h^{res}}{RT} = -T \left(\frac{\partial \bar{a}^{res}}{\partial T} \right)_{\rho, x_i} + (Z - 1) \quad (A.44)$$

$$875 \quad h = h^{res} + h^{id} \quad (A.45)$$

876

877 **16) Heat capacities [55]**

878 Heat capacities are computed as the sum of the ideal contribution [56] and the
879 correction terms calculated with the PC-SAFT EoS [54]. where C_p and C_v are the heat
880 capacities at constant pressure and volume respectively.

$$881 \quad C_v = C_{v,id} + C_{v,res} \quad (A.46)$$

$$882 \quad C_p = C_{p,id} + C_{p,res} \quad (A.47)$$

$$883 \quad C_v^{res} = -RT \left[2 \left(\frac{\partial \bar{a}^{res}}{\partial T} \right)_{\rho, x_i} + T \left(\frac{\partial^2 \bar{a}^{res}}{\partial T^2} \right)_{\rho, x_i} \right] \quad (A.48)$$

$$884 \quad C_p^{res} = C_v^{res} + R \frac{\left[\rho_m T \left(\frac{\partial^2 \bar{a}^{res}}{\partial \rho_m \partial T} \right)_{x_i} + \rho_m \left(\frac{\partial \bar{a}^{res}}{\partial \rho_m} \right)_{T, x_i} + 1 \right]^2}{\left[\rho_m^2 \left(\frac{\partial^2 \bar{a}^{res}}{\partial \rho_m^2} \right)_{T, x_i} + 2 \rho_m \left(\frac{\partial \bar{a}^{res}}{\partial \rho_m} \right)_{T, x_i} + 1 \right]} \quad (A.49)$$

885

886 **17) Speed of sound [55]**

887 The speed of sound is computed as:

$$888 \quad c = \sqrt{\frac{C_p}{C_v} \left(\frac{\partial P}{\partial \rho_m} \right)_T} \quad (A.50)$$

889

890 The derivatives needed to compute the speed of sound are:

$$891 \quad \left(\frac{\partial P}{\partial \rho_m} \right)_{T, x_i} = \left(\frac{\partial P}{\partial \eta} \right)_{T, x_i} \left(\frac{\partial \eta}{\partial \rho_m} \right)_{T, x_i} \quad (A.51)$$

892
$$\left(\frac{\partial \eta}{\partial \rho_m}\right)_{T,x_i} = \frac{\pi}{6} (\sum_i x_i m_i d_i^3) \quad (\text{A.52})$$

893
$$\left(\frac{\partial P}{\partial \eta}\right)_{T,x_i} = k_B T (10^{10})^3 \left[\rho_m \left(\frac{\partial Z}{\partial \eta}\right)_{T,x_i} + Z \left(\frac{\partial \rho_m}{\partial \eta}\right)_{T,x_i} \right] \quad (\text{A.53})$$

894
$$\left(\frac{\partial \rho_m}{\partial \eta}\right)_{T,x_i} = \frac{6}{\pi} (\sum_i x_i m_i d_i^3)^{-1} \quad (\text{A.54})$$

895
$$\left(\frac{\partial Z}{\partial \eta}\right)_{T,x_i} \text{ can be found in [57].} \quad (\text{A.55})$$

896

897 **18) Compute derivative internal energy respect temperature at constant density [55]**

898

899
$$\left(\frac{\partial e^{res}}{\partial T}\right)_{\rho,x_i} = -RT \left[2 \left(\frac{\partial \tilde{a}^{res}}{\partial T}\right)_{\rho,x_i} + \left(\frac{\partial^2 \tilde{a}^{res}}{\partial T^2}\right)_{\rho,x_i} * T \right] \quad (\text{A.56})$$

900
$$\frac{de^{id}}{dT} = C_v^{id} \quad (\text{A.57})$$

901

902
$$\left(\frac{\partial e}{\partial T}\right)_{\rho,x_i} = \left(\frac{\partial e^{res}}{\partial T}\right)_{\rho,x_i} + \frac{de^{id}}{dT} \quad (\text{A.58})$$

903

904 **19) Compute the new temperature using the Newton method**

905

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926 **Appendix 4 (Thermodynamic solver for mixtures at an unknown state)**

927 The pressure and the temperature are iterated employing a multidimensional Newton method
 928 until the density and the internal energy obtained in the PC-SAFT are the ones obtained from
 929 the conservative variables. The initial values of the pressure and the temperature are the ones
 930 already stored in the cell that is being solved.
 931

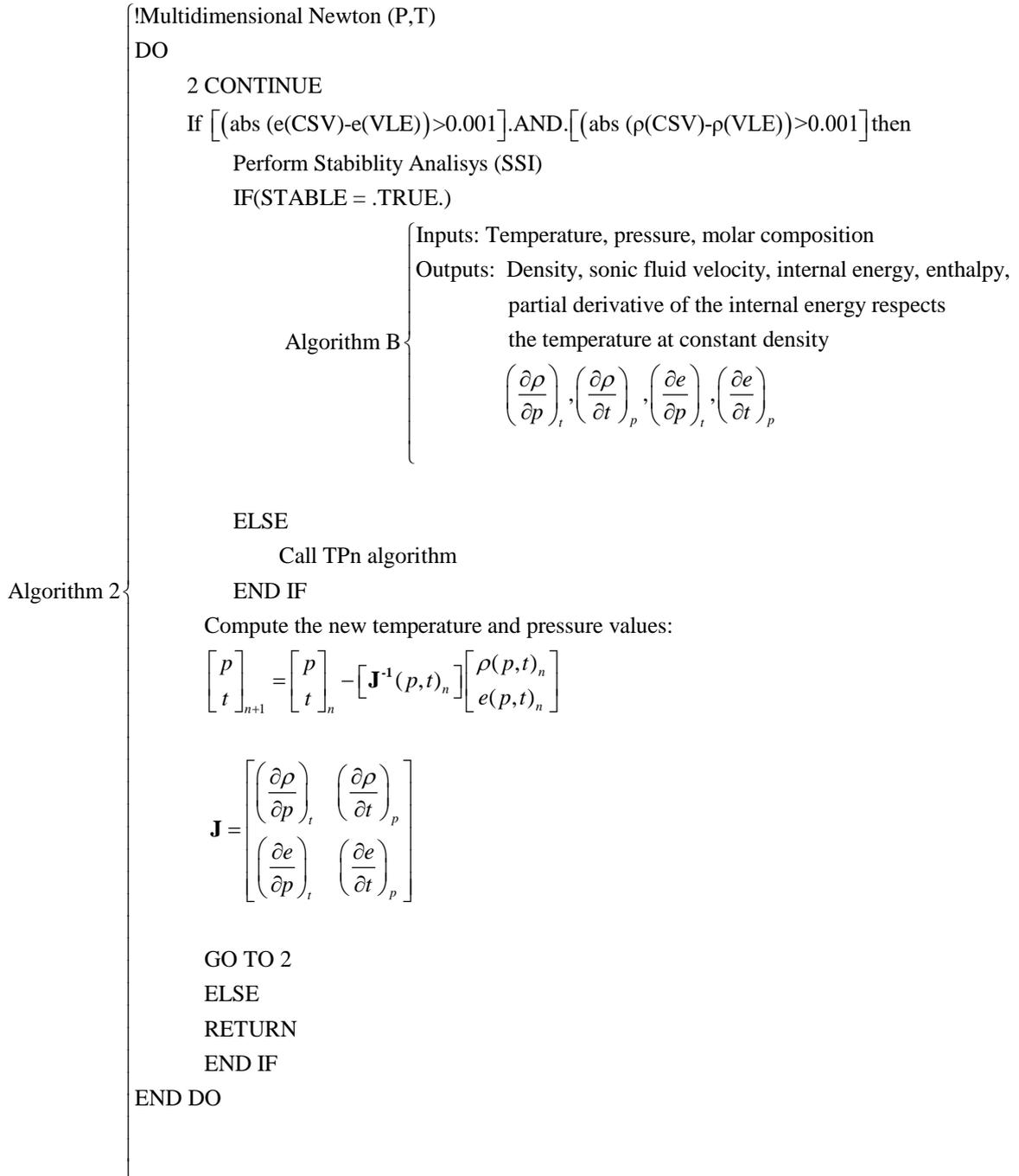


Figure A3. Schematic representation of the Algorithm 2

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 933
 934
 935
 936

937 **Appendix 5 (Algorithm B)**

938 **Inputs:** Temperature, pressure, molar composition.

939 **Output:** Density, speed of sound, internal energy, enthalpy, fugacities, partial derivative of the
 940 internal energy respect the temperature at constant pressure, partial derivative of the internal
 941 energy respect the pressure at constant temperature, partial derivative of the density respects
 942 the temperature at constant pressure and partial derivative of the density respect the temperature
 943 at constant pressure.

944 This algorithm is applied when the pressure and the temperature are iterated employing a
 945 multidimensional Newton method until the density and the internal energy computed are the
 946 ones obtained from the conservative variables.

947

$$948 \begin{bmatrix} p \\ t \end{bmatrix}_{n+1} = \begin{bmatrix} p \\ t \end{bmatrix}_n - [J^{-1}(p, t)_n] \begin{bmatrix} \rho(p, t)_n \\ e(p, t)_n \end{bmatrix} \quad (\text{A.59})$$

949

950 where

$$951 J = \begin{bmatrix} \left(\frac{\partial \rho}{\partial p}\right)_t & \left(\frac{\partial \rho}{\partial t}\right)_p \\ \left(\frac{\partial e}{\partial p}\right)_t & \left(\frac{\partial e}{\partial t}\right)_p \end{bmatrix} \quad (\text{A.60})$$

952

953

954 The independent variables of the PC-SAFT are the temperature and the density. Thus, it is
 955 necessary to perform the following transformations to obtain the partial derivatives needed for
 956 the multidimensional Newton method.

957

$$958 \left(\frac{\partial \rho}{\partial p}\right)_T = \left(\frac{\partial p}{\partial \rho}\right)_T^{-1} \quad \text{Reciprocity} \quad (\text{A.61})$$

$$959 \left(\frac{\partial \rho}{\partial t}\right)_p = -\left(\frac{\partial p}{\partial t}\right)_\rho \left(\frac{\partial p}{\partial \rho}\right)_T^{-1} \quad \text{Chain rule} \quad (\text{A.62})$$

$$960 \left(\frac{\partial e}{\partial p}\right)_T = \left(\frac{\partial e}{\partial \rho}\right)_T \left(\frac{\partial p}{\partial \rho}\right)_T^{-1} \quad \text{Chain rule} \quad (\text{A.63})$$

$$961 \left(\frac{\partial e}{\partial t}\right)_p = \left(\frac{\partial e}{\partial t}\right)_\rho - \left(\frac{\partial e}{\partial \rho}\right)_T \left(\frac{\partial \rho}{\partial t}\right)_p \left(\frac{\partial p}{\partial \rho}\right)_T^{-1} \quad \text{Triple product rule} \quad (\text{A.64})$$

962

963 The partial derivatives needed then are:

$$964 \left(\frac{\partial p}{\partial t}\right)_\rho, \left(\frac{\partial p}{\partial \rho}\right)_T, \left(\frac{\partial e}{\partial \rho}\right)_T, \left(\frac{\partial e}{\partial t}\right)_\rho$$

965

966 **Steps**

967 1) **Compute temperature-dependent segment diameter d of component i (A.12)**

968 2) **Compute mean segment number (A.13)**

969 3) **Compute radial distribution function of the hard-sphere fluid (A.14)**

970 4) **Reduce density iterative method**

971 a. $\eta_{INIT} = 0.45$

972 b. **Compute contribution of the hard sphere to the compressibility factor**
 973 **(A.16)**

974 c. **Compute hard-chain contribution to the compressibility factor (A.17)**

975

976

Algorithm B

- 1) Compute segment diameter of each component (A.12)
- 2) Compute mean segment number (A.13)
- 3) Compute radial distribution function of the hard sphere fluid (A.14)
- 4) Reduce density iterative method
 - $\eta_{INIT} = 0.45$
 - DO
 - If $[(abs(p^{calc} - p^{sys}) > 0.001)]$ then
 - Compute hard sphere fluid contribution to the compressibility factor (A.16)
 - Compute hard chain contribution to the compressibility factor (A.17)
 - Compute dispersion contribution to the compressibility factor (A.19)
 - Compute total compressibility factor (A.30)
 - Compute pressure (A.31)
 - Derivative of pressure respect reduce density
 - Compute new pressure using Newton method
 - END IF
 - END DO
- 5) Compute the contribution of the hard-sphere system to the residual Helmholtz free energy temperature derivative (A.32)
- 6) Compute the hard-chain reference contribution to the residual Helmholtz free energy temperature derivative (A.35)
- 7) Compute the dispersion contribution to the residual Helmholtz free energy temperature derivative (A.37)
- 8) Compute the residual Helmholtz free energy temperature derivative (A.41)
- 9) Compute internal energy (A.43)
- 10) Compute enthalpy (A.45)
- 11) Compute heat capacities (A.46,A.47)
- 12) Compute speed of sound (A.50)
- 13) Compute the derivatives of the Helmholtz free energy residual contribution of the hard-sphere system respect the molar fraction of the components (A.66)
- 14) Compute the derivative of the hard-chain reference contribution to the residual Helmholtz free energy respect the molar fraction of the components (A.68)
- 15) Compute derivative of the dispersion contribution to the residual Helmholtz free energy respect the molar fraction of the components (A.70)
- 16) Compute the chemical potential (A.78)
- 17) Compute the fugacity coefficient (A.79)
- 18) Compute the partial derivative of the pressure respect the density at constant temperature (A.80)
- 19) Compute the partial derivative of the pressure respect the temperature at constant density (A.81)
- 20) Compute the partial derivative of the internal energy respect the temperature at constant density (A.58)
- 21) Compute the partial derivative of the internal energy respect the density at constant temperature (A.84)
- 22) Compute the partial derivative of the density respect the pressure at constant temperature (A.61)
- 23) Compute the partial derivative of the density respect the temperature at constant pressure (A.62)
- 24) Compute the partial derivative of the internal energy respect the pressure at constant temperature (A.63)
- 25) Compute the partial derivative of the internal energy respect the temperature at constant pressure (A.64)

Algorithm B

977

978

Figure A4 Schematic representation of the Algorithm B

979

980

- 981 **d. Compute dispersion contribution to the compressibility factor (A.19)**
 982 **e. Compute compressibility factor (A.30)**
 983 **f. Compute pressure (A.31)**
 984 **g. Derivative of pressure respect reduce density**
 985

$$986 \quad \left(\frac{\partial P}{\partial \eta}\right) = \left[\rho_m \left(\frac{\partial Z}{\partial \eta}\right) + Z \left(\frac{\partial \rho_m}{\partial \eta}\right)\right] (10^{10})^3 Z k_B T \quad (A.65)$$

987

988 **h. Compute the new pressure using the Newton method**

- 989 **5) Compute temperature derivative of the Helmholtz free energy residual**
 990 **contribution of the hard-sphere system (A.32)**
 991 **6) Compute temperature derivative of the hard-chain reference contribution to the**
 992 **residual Helmholtz free energy (A.35)**
 993 **7) Compute temperature derivative of the dispersion contribution to the residual**
 994 **Helmholtz free energy (A.37)**
 995 **8) Compute temperature derivative of the Helmholtz free energy (A.41)**
 996 **9) Compute the internal energy (A.43)**
 997 **10) Compute enthalpy (A.45)**
 998 **11) Compute heat capacities (A.46-47)**
 999 **12) Compute speed of sound (A.50)**
 1000 **13) Compute the derivatives of the Helmholtz free energy residual contribution of the**
 1001 **hard-sphere system respect the molar fraction of the components.**
 1002

$$1003 \quad \left(\frac{\partial \tilde{a}^{hs}}{\partial x_k}\right)_{T,\rho,x_{j \neq k}} = -\frac{\zeta_{0,xk}}{\zeta_0} \tilde{a}^{hs} + \frac{1}{\zeta_0} \left[\frac{3(\zeta_{1,xk}\zeta_2 + \zeta_1\zeta_{2,xk})}{(1-\zeta_3)} + \frac{3\zeta_1\zeta_2\zeta_{3,xk}}{(1-\zeta_3)^2} + \frac{3\zeta_2^2\zeta_{2,xk}}{\zeta_3(1-\zeta_3)^2} + \right. \\ 1004 \quad \left. \frac{\zeta_2^3\zeta_{3,xk}(3\zeta_3-1)}{\zeta_3^2(1-\zeta_3)^3} + \left(\frac{3\zeta_2^2\zeta_{2,xk}\zeta_3 - 2\zeta_2^3\zeta_{3,xk}}{\zeta_3^3} - \zeta_{0,xk} \right) \ln(1-\zeta_3) + \left(\zeta_0 - \frac{\zeta_2^2}{\zeta_3^2} \right) \frac{\zeta_{3,xk}}{(1-\zeta_3)} \right] \quad (A.66)$$

1005

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where

$$1007 \quad \zeta_{n,xk} = \left(\frac{\partial \zeta_n}{\partial x_k}\right)_{T,\rho,x_{j \neq k}} = \frac{\pi}{6} \rho m_k (d_k)^n \quad (A.67)$$

1008

- 1009 **14) Compute the derivative of the hard-chain reference contribution to the residual**
 1010 **Helmholtz free energy respect the molar fraction of the components.**

$$1011 \quad \left(\frac{\partial \tilde{a}^{hc}}{\partial x_k}\right)_{T,\rho,x_{j \neq k}} \\ 1012 \quad = m_k \tilde{a}^{hs} + \tilde{m} \left(\frac{\partial \tilde{a}^{hs}}{\partial x_k}\right)_{T,\rho,x_{j \neq k}} - \sum_i x_i (m_i - 1) (g_{ii}^{hs})^{-1} \left(\frac{\partial g_{ii}^{hs}}{\partial x_k}\right)_{T,\rho,x_{j \neq k}} \quad (A.68)$$

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where

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$$1017 \quad \left(\frac{\partial g_{ij}^{hs}}{\partial x_k}\right)_{T,\rho,x_{j \neq k}} = \frac{\zeta_{3,xk}}{(1-\zeta_3)^2} + \left(\frac{d_i d_j}{d_i + d_j}\right) \left(\frac{3\zeta_{2,xk}}{(1-\zeta_3)^2} + \frac{6\zeta_2\zeta_{3,xk}}{(1-\zeta_3)^3}\right) \\ 1018 \quad + \left(\frac{d_i d_j}{d_i + d_j}\right)^2 \left(\frac{4\zeta_2\zeta_{2,xk}}{(1-\zeta_3)^3} + \frac{6\zeta_2^2\zeta_{3,xk}}{(1-\zeta_3)^4}\right) \quad (A.69)$$

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15) Compute derivative of the dispersion contribution to the residual Helmholtz free energy respect the molar fraction of the components.

$$\begin{aligned} \left(\frac{\partial \bar{a}^{disp}}{\partial x_k}\right)_{T,\rho,x_{j \neq k}} &= -2\pi\rho \left[I_{1,xk} \overline{m^2 \varepsilon \sigma^3} + I_1 \overline{(m^2 \varepsilon \sigma^3)}_{xk} \right] - \\ &\pi\rho \left\{ \left[m_k C_1 I_2 + \bar{m} C_{1,xk} I_2 + \bar{m} C_1 I_{2,xk} \right] \overline{m^2 \varepsilon \sigma^3} + \right. \\ &\left. \bar{m} C_1 I_2 \overline{(m^2 \varepsilon \sigma^3)}_{xk} \right\} \end{aligned} \quad (A.70)$$

where

$$\overline{(m^2 \varepsilon \sigma^3)}_{xk} = 2m_k \sum_j x_j m_j \left(\frac{\varepsilon_{kj}}{kT}\right) \sigma_{kj}^3 \quad (A.71)$$

$$\overline{(m^2 \varepsilon^2 \sigma^3)}_{xk} = 2m_k \sum_j x_j m_j \left(\frac{\varepsilon_{kj}}{kT}\right)^2 \sigma_{kj}^3 \quad (A.72)$$

$$C_{1,xk} = C_2 \zeta_{3,xk} - C_1^2 \left[m_k \frac{8\eta - 2\eta^2}{(1-\eta)^4} - m_k \frac{20\eta - 27\eta^2 + 12\eta^3 - 2\eta^4}{[(1-\eta)(2-\eta)]^2} \right] \quad (A.73)$$

$$I_{1,xk} = \sum_{i=0}^6 [a_i(\bar{m}) i \zeta_{3,xk} \eta^{i-1} + a_{i,xk} \eta^i] \quad (A.74)$$

$$I_{2,xk} = \sum_{i=0}^6 [b_i(\bar{m}) i \zeta_{3,xk} \eta^{i-1} + b_{i,xk} \eta^i] \quad (A.75)$$

$$a_{i,xk} = \frac{m_k}{\bar{m}^2} a_{1i} + \frac{m_k}{\bar{m}^2} \left(3 - \frac{4}{\bar{m}}\right) a_{2i} \quad (A.76)$$

$$b_{i,xk} = \frac{m_k}{\bar{m}^2} b_{1i} + \frac{m_k}{\bar{m}^2} \left(3 - \frac{4}{\bar{m}}\right) b_{2i} \quad (A.77)$$

16) Compute the chemical potential.

$$\frac{\mu_k^{res}(T,v)}{kT} = \bar{a}^{res} + (Z - 1) + \left(\frac{\partial \bar{a}^{res}}{\partial x_k}\right)_{T,v,x_{i \neq j}} - \sum_{j=1}^N \left[x_j \left(\frac{\partial \bar{a}^{res}}{\partial x_j}\right)_{T,v,x_{i \neq j}} \right] \quad (A.78)$$

17) Compute the fugacity coefficient.

$$\ln \varphi_k = \frac{\mu_k^{res}(T,v)}{kT} - \ln Z \quad (A.79)$$

18) Compute the partial derivative of the pressure respect the density at constant temperature.

$$\left(\frac{\partial P}{\partial \rho_m}\right)_T = k_B T (10^{10})^3 \left[\left(\frac{\partial Z}{\partial \rho_m}\right)_T \rho_m + Z \right] \quad (A.80)$$

19) Compute the partial derivative of the pressure respect the temperature at constant density.

$$\left(\frac{\partial P}{\partial T}\right)_\rho = k_B (10^{10})^3 \rho_m \left[\left(\frac{\partial Z}{\partial T}\right)_\rho T + Z \right] \quad (A.81)$$

20) Compute the partial derivative of the internal energy respect the temperature at constant density (A.58)

21) Compute the partial derivative of the internal energy respect the density at constant temperature.

$$\left(\frac{\partial e^{res}}{\partial \rho_m}\right)_T = -RT^2 \left(\frac{\partial \bar{a}^{res}}{\partial T \partial \rho_m}\right)_T \quad (A.82)$$

1057 $\frac{de^{id}}{d\rho_m} = 0.0$ (A.83)

1058

1059 $\left(\frac{\partial e}{\partial T}\right)_{\rho, x_i} = \left(\frac{\partial e^{res}}{\partial T}\right)_{\rho, x_i}$ (A.84)

1060

1061 **22) Compute the partial derivative of the density respect the pressure at constant**
1062 **temperature (A.61)**

1063 **23) Compute the partial derivative of the density respect the temperature at constant**
1064 **pressure (A.62)**

1065 **24) Compute the partial derivative of the internal energy respect the pressure at**
1066 **constant temperature (A.63)**

1067 **25) Compute the partial derivative of the internal energy respect the temperature at**
1068 **constant pressure (A.64)**

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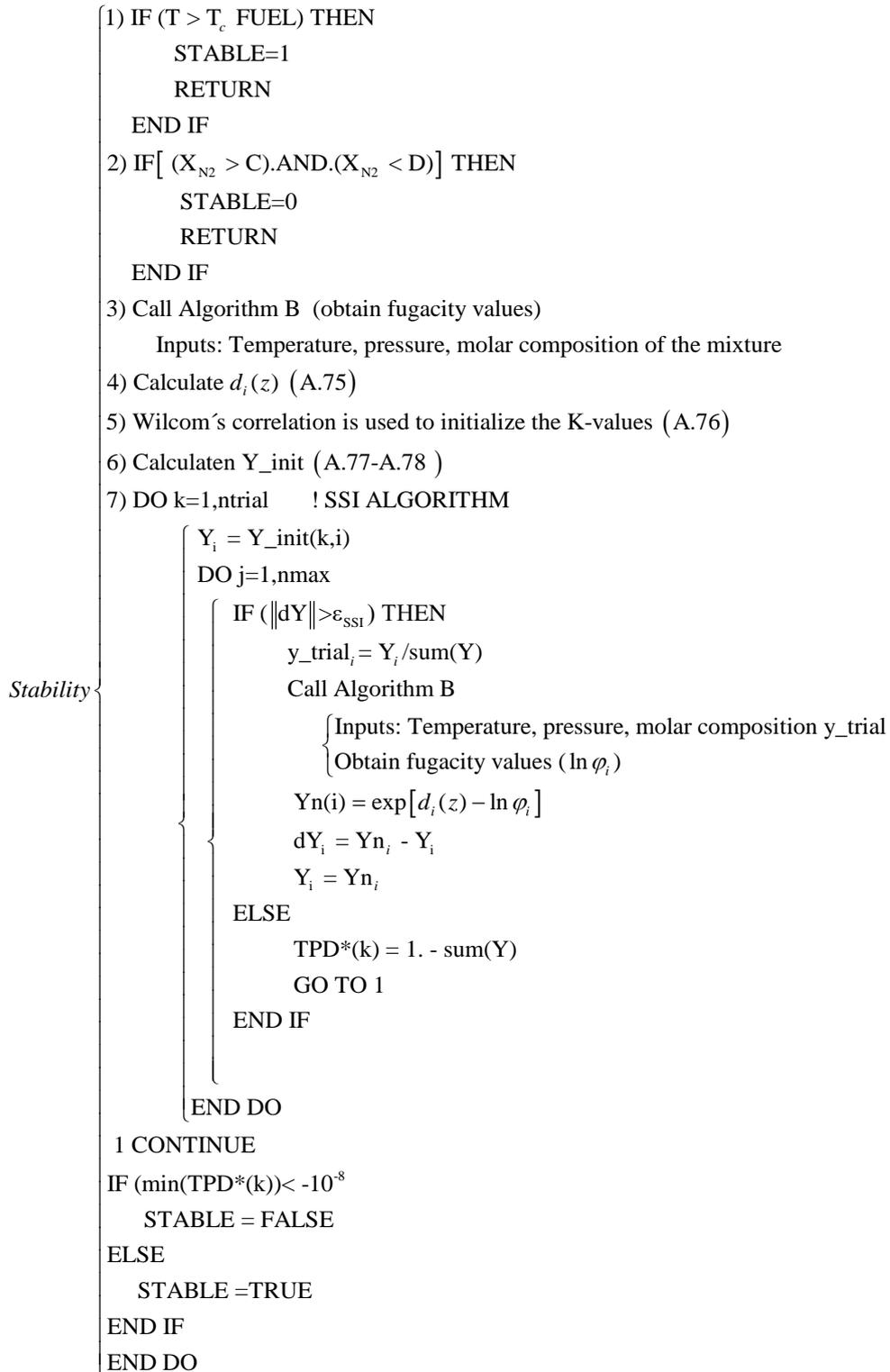
1098 **Appendix 6 (Stability)**

1099 **Input:** Temperature, pressure and molar composition of the mixture.

1100 **Output:** To know if the mixture is stable or not (one or two phases).

1101 The Successive Substitution Iteration (SSI) algorithm ([16], [46]) (without the Newton
1102 method) has been employed.

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Figure A5 Schematic representation of the stability algorithm

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Steps

- 1) **The mixture is stable if the temperature is higher than T_c Fuel (STABLE = 1)**
Any mixture with a temperature higher than the fuel critical temperature will not be in a VLE state (STABLE = 1). This kind of filters are applied to reduce the computational time.
- 2) **The mixture is unstable if the nitrogen molar fraction is bigger than C and lower than D (STABLE = 0)**
The coefficients B and C are case dependent. For example, by performing an injection of n-dodecane at 363K in a combustion chamber at 900K, the nitrogen mole fraction at which the fuel starts vaporizing depends on the pressure in the combustion chamber. Considering Diesel engines at high-load operation conditions (11MPa) it would be safe to consider that any mixture with a nitrogen molar fraction bigger than 0.35 and lower than 0.7 will be in a VLE state, see Figure 4.
- 3) **Call Algorithm B to obtain fugacity coefficient values $\ln\phi_i(z_i)$**
Inputs: Temperature, pressure, molar composition of the mixture
- 4) **Calculate $d_i(z)$**
$$d_i(z) = \ln\phi_i(z_i) + \ln z_i \quad (\text{A.85})$$
- 5) **The Wilcom's correlation is used to initialize the K-values**
$$K_i = \frac{p_{ci}}{p} \exp \left[5.37(1.0 + w_i) \left(1.0 - \frac{T_{ci}}{T} \right) \right] \quad (\text{A.86})$$

being
$$K_i = \frac{x_i}{y_i}$$

where p_{ci} is the critical pressure of the component i, T_{ci} is the critical temperature of the component i, w_i is the acentric factor of the component i.
- 6) **Calculate trial phases Y (two trials)**
For the trial 1:
$$Y(1, comp) = \begin{cases} \frac{z_i}{K_i} & (\text{Liquid phase}) \\ z_i K_i & (\text{Vapor phase}) \end{cases} \quad (\text{A.87})$$

For the trial 2:
$$Y(2, comp) = \begin{cases} \frac{z_i}{K_i^3} & (\text{Liquid phase}) \\ \frac{z_i K_i}{3} & (\text{Vapor phase}) \end{cases} \quad (\text{A.88})$$
- 7) **SSI-Algorithm described in Figure A5.**

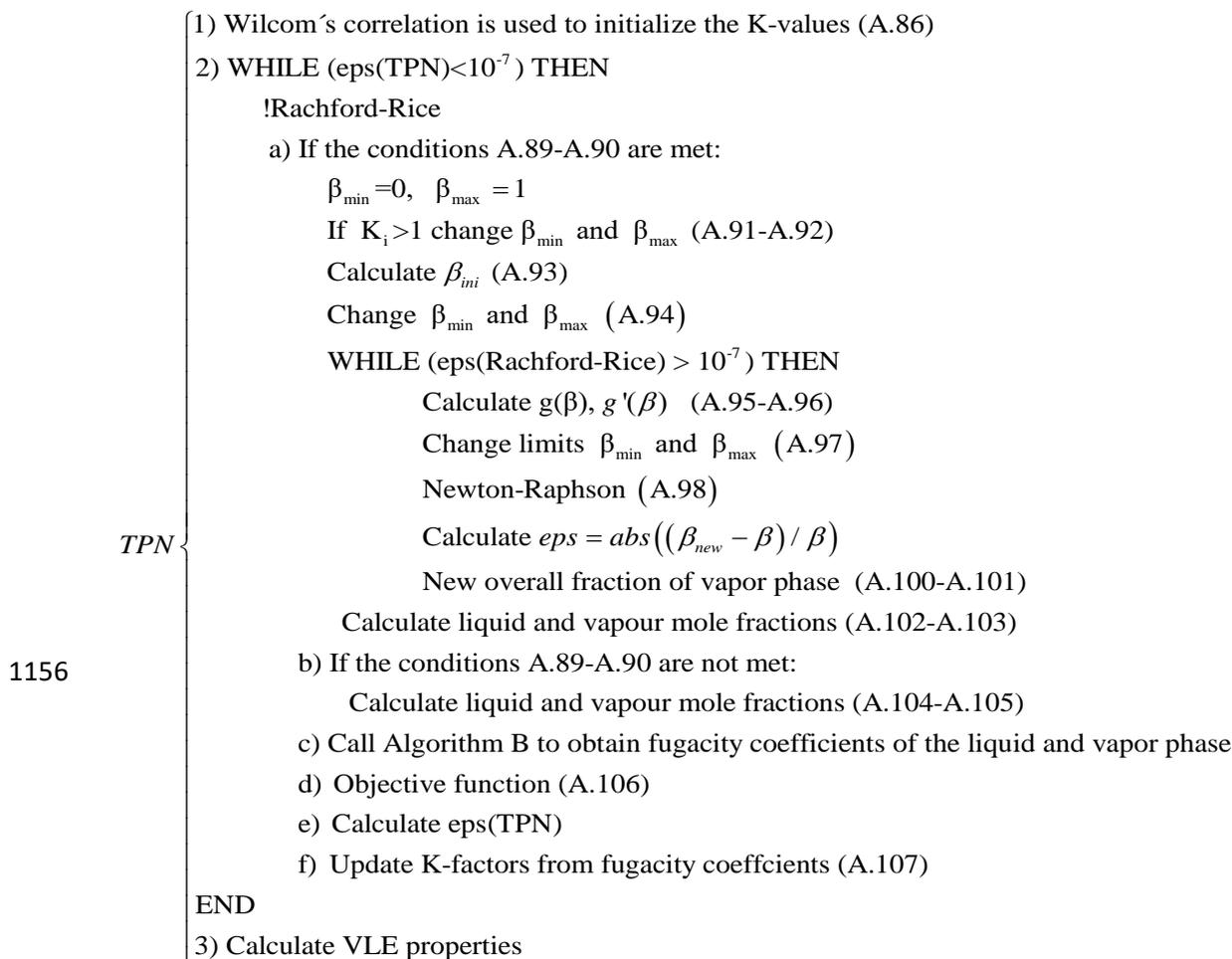
1149 **Appendix 7 (TPn Algorithm)**

1150 **Input:** Temperature, pressure and molar composition of the mixture.

1151 **Output:** Density of the mixture, internal energy of the mixture, speed of sound of the mixture
1152 and enthalpy of the mixture.

1153 This algorithm is employed to to perform equilibrium calculations at specified temperature,
1154 pressure and overall composition. A successive substitution method is employed [16], [44].

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Figure A6 Schematic representation of the TPn algorithm

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1159 **1) The Wilcom's correlation is used to initialize the K-values (A.86)**

1160

1161 **2) WHILE (eps(TPN)<10⁻⁷) THEN**

1162

1163 **Solve Rachford-Rice**

1164 **a. Check conditions A.79-A.80 to know if there is a solution in the interval**
1165 **beta [0,1]. If the conditions are met set beta_{min}=0, beta_{max} = 1. If not, go to step**

1166

2.b.

1167
$$\sum_{i=1}^C z_i K_i - 1 > 0 \quad (A.89)$$

1168

1169
$$1 - \sum_{i=1}^C \frac{z_i}{K_i} < 0 \quad (A.90)$$

1170

1171

- **If $K_i > 1$ then**

1172

$$\beta_{\min} = \max_i \left[0, \frac{K_i z_i - 1}{K_i - 1} \right] \quad (\text{A.91})$$

1173

$$\beta_{\max} = \min_i \left[1, \frac{1 - z_i}{1 - K_i} \right] \quad (\text{A.92})$$

1174

1175

- **Calculate $\beta_{ini} = 0.5(\beta_{\min} + \beta_{\max})$** (A.93)

1176

1177

- **Change limits**

1178

$$g(\beta_{ini}) > 0 \rightarrow \beta_{\min} = \beta_{ini},$$

1179

$$g(\beta_{ini}) < 0 \rightarrow \beta_{\max} = \beta_{ini} \quad (\text{A.94})$$

1180

1181

- **WHILE (eps(Rachford-Rice) $> 10^{-7}$) then**

1182

1183

- **Calculate $g(\beta)$, $g'(\beta)$**

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$$g(\beta) = \sum_{i=1}^C (y_i - x_i) = \sum_{i=1}^C \frac{z_i(K_i - 1)}{1 - \beta + \beta K_i} = 0 \quad (\text{A.95})$$

1185

$$g'(\beta) = - \sum_{i=1}^C \frac{z_i(K_i - 1)^2}{(1 - \beta + \beta K_i)^2} < 0 \quad (\text{A.96})$$

1186

1187

- **Change limits**

1188

$$g > 0 \rightarrow \beta_{\min} = \beta,$$

1189

$$g < 0 \rightarrow \beta_{\max} = \beta \quad (\text{A.97})$$

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- **Newton-Raphson**

1192

$$\Delta\beta = - \frac{g(\beta)}{dg/d\beta}$$

1193

$$\beta_{new} = \beta + \Delta\beta \quad (\text{A.98})$$

1194

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- **Calculate eps**

1196

$$eps = \text{abs}((\beta_{new} - \beta)/\beta) \quad (\text{A.99})$$

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- **New overall fraction of vapor phase:**

1199

$$\beta = \beta_{new} \text{ if } \beta_{new} \text{ is inside the interval } [\beta_{\min}, \beta_{\max}] \quad (\text{A.100})$$

1200

1201

- **If it is not, it is calculated as:**

1202

$$\beta = 0.5(\beta_{\min} + \beta_{\max}), \quad \beta_{\min} < \beta_{new} < \beta_{\max} \quad (\text{A.101})$$

1203

1204

- **Calculate liquid and vapour mole fractions**

1205

$$x_i = \frac{z_i}{1 - \beta + \beta K_i} \quad (\text{A.102})$$

1206

$$y_i = \frac{K_i z_i}{1 - \beta + \beta K_i} \quad (\text{A.103})$$

1207

1208

- b. If the conditions A.89-A.90 are not met [58]:**

1209

- If $\sum z_i / K_i \leq 1$ the liquid and vapour mole fractions are computed as:**

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$$\beta = 1$$

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$$x_i = z_i / K_i$$

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$$y_i = z_i \quad (\text{A.104})$$

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Normalization of x_i

If $\sum z_i K_i \leq 1$ the liquid and vapour mole fractions are computed as:

$$\begin{aligned} \beta &= 0 \\ x_i &= z_i \\ y_i &= z_i * K_i \end{aligned} \tag{A.105}$$

Normalization of y_i

c. Call Algorithm B to obtain fugacity coefficients of the liquid and vapor phase

Inputs: Temperature, pressure, molar composition of the liquid or vapor

d. Objective function

$$F_i = \ln \varphi_v(T, p, y) - \ln \varphi_l(T, p, x) + \ln K_i = 0 \tag{A.106}$$

e. Calculate eps(TPN)

f. Update K-factors from fugacity coefficients

$$K_i = \exp(\ln \varphi_l(T, p, x) - \ln \varphi_v(T, p, y)) \tag{A.107}$$

3) Compute VLE properties

a. The phase fraction on mass (β_m) basis is computed as:

$$\beta_m = \beta \frac{M_V(\text{mixture molar mass in liquid phase})}{M_T(\text{mixture molar mass})} \tag{A.108}$$

b. The equilibrium volume (v_{EQ}) is computed as:

$$v_{EQ} = \beta_m v_v + (1 - \beta_m) v_l \tag{A.109}$$

c. The equilibrium density is computed as:

$$\rho = \frac{1}{v_{EQ}} \tag{A.110}$$

d. The equilibrium internal energy is computed as:

$$e = \beta_m e_v + (1 - \beta_m) e_l \tag{A.111}$$

e. The equilibrium enthalpy is computed as:

$$h = \beta_m h_v + (1 - \beta_m) h_l \tag{A.112}$$

f. The speed of sound in the VLE state was computed using Wallis formula:

$$\frac{1}{\rho c_{wallis}^2} = \frac{\theta}{\rho_v c_v^2} + \frac{1-\theta}{\rho_l c_l^2} \tag{A.113}$$

where the vapour volume fraction (θ) is computed as:

$$\theta = \frac{\rho - \rho_v}{\rho_v - \rho_l} \tag{A.114}$$

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