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# 1 Aeroacoustic investigation of a ducted wind turbine employing bio- 2 inspired airfoil profiles

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10

## 11 Abstract

12 Ducted wind turbines for residential purposes are characterized by a lower diameter with  
13 respect to conventional wind turbines for on-shore applications. The noise generated by the  
14 rotor plays a significant role in the overall aerodynamic noise. By making modifications to the  
15 blade sections of the wind turbine, we can alter the contributions of aeroacoustic noise sources.  
16 This study introduces innovative wind turbine blade designs inspired by owl wing  
17 characteristics, achieving significant noise reduction without compromising aerodynamic  
18 performance. A three-dimensional (3D) scan of an owl wing was first employed to derive a  
19 family of airfoils. The airfoils were employed to modify the blade of a referenced wind turbine  
20 airfoil section at various positions on the blade span to determine a blade operating more  
21 efficiently at the tip-speed ratio of the original one. While maintaining the same aerodynamic  
22 performance, the bio-inspired profiles show a more uniform pressure coefficient distribution,  
23 considerably decreasing in the noise level. Furthermore, this study makes considerable  
24 progress in ducted wind turbine design by obtaining an 8 dB noise reduction and a 12%  
25 improvement in sound pressure level. An in-depth aerodynamic examination shows a 6.4% rise  
26 in thrust force coefficient and optimized power coefficients, reaching a peak at a Tip Speed  
27 Ratio (TSR) of 8, demonstrating improved energy conversion efficiency. The results highlight  
28 the dual advantage of the innovative design: significant noise reduction and enhanced  
29 aerodynamic efficiency, offering a promising alternative for urban wind generation.  
30

31 **Keywords:** Ducted Wind Turbine, Aeroacoustics, Barn Owl, Large Eddy Simulation.

## 32 Nomenclatures

AOA	Angle of Attack	$C_T$	thrust force coefficient
$c$	Chord of Airfoil	TSR	Tip Speed Ratio
$C_l$	Lift Coefficient	SPL	Sound Pressure Level
$C_d$	Drag Coefficient	OASPL	Overall Sound Pressure Level
$C_l/C_d$	Lift-to-drag ratio	$C_{PWR}$	Power Coefficient
2D / 3D	Two / Three Dimensional	LES	Large Eddy Simulation
WT	Wind Turbine	FW-H	Ffowcs Williams-Hawkings
DWT	Ducted Wind Turbine	CFD	Computational Fluid Dynamics

### 33 Introduction

34 The effect of rising global warming has shifted the world to clean energy sources; the most  
 35 efficient source of clean energy is wind energy [1]. The use of such renewable energy has  
 36 grown rapidly [2], to the point of justifying the usage of wind turbines in profitable locations  
 37 in urban areas, to reduce the costs of energy delivery to the user. The integration in the urban  
 38 environment comes at the cost of reducing the rotor sizes of the wind turbine, by adding a duct  
 39 to increase the incoming flow speed lowering the effect of incoming turbulent fluctuations, and  
 40 having to respect more stringent aeroacoustic regulations [3].

41 The reduction of the wind turbine rotor changes the contributions of the aeroacoustic noise  
 42 sources, since the turbine is now operating at a relatively lower Reynolds number and at a much  
 43 higher rotational speed than a conventional one. In this respect, the loading and thickness  
 44 contributions are no longer negligible. The overall aerodynamic noise is a combination of the  
 45 one produced by the rotor and the one determined by the duct. While a lot of studies have  
 46 focused on the coupling between the duct or diffuser and the rotor, only a few have proposed  
 47 the use of specific airfoils for such low Reynolds number and high turbulence applications.  
 48 Bio-inspired airfoils have been shown to possess particular characteristics to allow birds to fly  
 49 in very turbulent environments, with an extremely high lift-to-drag ratio, and at the same time  
 50 do silently [4]. This has inspired the use of such airfoils for drones and small rotor applications.  
 51 Despite being known for their feather characteristics, owl's wings have an additional  
 52 characteristic. The high lift-to-drag ratio of the profile, allows the bird to enormously reduce  
 53 the flying speed to sustain the bird's weight. Combined with the additional serrated leading  
 54 edge, the velvety surface and the fringes at the trailing edge, the owl's wing performance is the  
 55 most silent in the animal kingdom in the final phase of attacking the prey [5-7]. From an  
 56 aerodynamic point of view the combination of the previous factors, seems to also produce a  
 57 more favorable and thinner boundary layer, which helps in increasing the aerodynamic  
 58 performance of the wing. Various studies have been performed experimentally [8, 9],  
 59 numerically [10-12] and in real flight [13] to use the owl wing characteristics.

60 In the realm of aerodynamics, flow and noise control are pivotal for enhancing the performance  
 61 and reducing the environmental impact of wind turbines. Two primary methods are employed  
 62 to achieve these objectives: active and passive flow control techniques. Active methods, such  
 63 as the use of dielectric barrier discharge plasma actuators demonstrated by Lee et al. [14],  
 64 actively manipulate the flow field around structures to control separation and reduce drag,  
 65 thereby influencing noise generation. On the other hand, passive methods involve structural  
 66 modifications to the body, which passively influence the flow and noise characteristics. An  
 67 example is the use of grooved surfaces on deflectors to improve the aerodynamic performance  
 68 of Savonius wind turbines, as explored by Fatahian et al. [15]. This research aligns with passive  
 69 flow control strategies by adopting a bio-inspired model that leverages the silent flight  
 70 characteristics of owl wings. The integration of airfoil profiles inspired by the natural wing  
 71 structure of owls represents a novel approach in the design of ducted wind turbines to passively  
 72 control flow and reduce noise. Such bio-inspired designs, as evidenced by the comparative  
 73 analysis of flow control over a circular cylinder with detached flexible and rigid splitter plates  
 74 by Eydi et al. [16], underscore the potential of nature-inspired solutions in engineering  
 75 applications. Our study builds upon this foundation, employing passive flow control through  
 76 bio-mimicry to achieve a harmonious balance between aerodynamic efficiency and noise  
 77 reduction in wind turbine design. Additionally, Song et al. [17] demonstrated that the bionic  
 78 edge design strategy can effectively control the turbulent flow field and effectively break down  
 79 airflow near the trailing edge. This leads to improved thrust and decreased noise levels.

80 According to the literature review in this study, it was concluded that very few studies focused  
81 on isolating the airfoil characteristics and employing them for industrial applications. In a study  
82 by Liu et al. [18], a laser scanner was used to scan the owl wing and characterize its geometric  
83 characteristics. These characteristics included camber, chord, twist, and thickness distribution.  
84 An analysis of flow interactions with surfaces and flow physics surrounding an owl airfoil was  
85 conducted by Klan et al. [19]. Bachmann et al. [20] provided a comprehensive characterization  
86 of the wings and feathers of barn owls in high spatial resolution. They conclude that barn owls  
87 have evolved specialized wings and feathers to reduce flight noise. To investigate the wings at  
88 high spatial resolution, they used confocal laser scanning microscopy, surface digitizing, and  
89 computed tomography. They found that these birds of prey have huge wings relative to their  
90 body mass, which enables slow flights with increased maneuverability. At low speeds, the  
91 researchers found that modifications to the wings' surfaces and edges helped stabilize airflow.  
92 Geyer et al. [21] conducted a comprehensive study on owl wings. In addition to performing  
93 numerical calculations, they conducted laboratory tests on the wings of owls and several other  
94 birds. According to their findings, an owl's wings can be 20 decibels quieter than other birds'  
95 wings when gliding. This was achieved by exploiting the relatively lower speed at which the  
96 owl's profile was able to operate. Kondo et al. [22] studied the aerodynamic characteristics of  
97 an owl-like airfoil at a Reynolds number of 2300. Their results indicate that the deeply concave  
98 lower surface of the owl-like airfoil contributes to lift augmentation, and both a round leading  
99 edge and a flat upper surface lead to lift enhancement and drag reduction, determined by the  
100 presence of a thin laminar separation bubble near the leading edge. Subsequently, the owl-like  
101 airfoil has a higher lift-to-drag ratio than the high lift-to-drag Ishii airfoil at low Reynolds  
102 number. A new airfoil for the wind turbine blades was designed and used in the wind turbine  
103 blades by Tian et al. [23]. Results show that the bio-inspired airfoil inspired by the Long-eared  
104 Owl's wing has a superior lift coefficient and stalling performance, and thus can enhance wind  
105 turbine blade performance. An owl inspired airfoil without serrations and a Downy wing  
106 surface was compared to a NACA airfoil at low Reynolds numbers by Anyoji et al. [24].  
107 According to their results, the owl inspired airfoil has more lift and generally performs better  
108 than the base airfoil at low Reynolds numbers. Moslem et al. [25] demonstrated that a  
109 bioinspired propeller not only diminishes both harmonic and broadband noise but also achieves  
110 a superior noise level compared to the baseline configuration. Aono et al. [26] investigated the  
111 aerodynamics of an owl wing-like airfoil for low Reynolds numbers using numerical methods  
112 and the LES turbulence model. They compared the simulation results with several conventional  
113 airfoils and showed that the owl wing-inspired airfoil has a higher lift-to-drag coefficient than  
114 the other airfoils compared. They reported that this increase was due to the creation of a high-  
115 pressure area in the suction area of the airfoil due to the curvature of the owl airfoil.  
116 Muthuramalingam et al. [27] explored laminar flow control by employing leading-edge  
117 serration, demonstrating a postponement in the transition from laminar to turbulent flow. This  
118 effect parallels the phenomenon observed in owl flight, contributing to further noise reduction.  
119 Despite all the previous results, whether these airfoils can be reliably applied for a rotating  
120 blade at low Reynolds number is still under debate. While in fact the aspect ratio is very similar,  
121 the loading distribution of a rotating blade is relatively different from the distribution of a bird's  
122 wing in gliding conditions.

123 In this manuscript, an investigation is made to study how bio-inspired airfoils could be  
124 employed to outperform the loading distribution of a rotating blade. The study proceeds by  
125 evaluating how the aerodynamic performance would affect the aeroacoustic footprint of the  
126 rotor, including loading and thickness noise and discussing the broadband part due to the  
127 change in boundary layer characteristics. The manuscript is organized as follows: In section 2  
128 of this research, a family of airfoils is produced through 3D scanning. These airfoils are then

utilized in the design of ducted wind turbine blades, as detailed in section 3. The following sections present a comprehensive numerical analysis of cases (benchmark and multi-section blades), along with the results of aerodynamic and aeroacoustic prediction.

### Airfoil family generation

To generate a family of bio-inspired airfoils, a taxidermy owl as shown in Figure 1 has been placed in the center of a 3D scanner. Circular targets mark the owl wing to allow for combining multiple fields of view in a unique 3D reconstruction. In this test, the Solutionix C500-Structured Light 3D Scanner has been used, with a final reconstruction accuracy of 0.01 mm. The center points of the wing are relatively less accurate, due to possible errors induced by the presence of the velvet surface and the feathers.

Illustrated below is the barn owl wing alongside its corresponding reverse model (Figure 2). The airfoil sections exhibit varying airfoil shapes and chord lengths. The data fitting procedure involved using MATLAB to employ Polynomial fitting. For both the upper and lower surfaces of the airfoil, an independent polynomial of degree six was selected. Figure 3 provides an illustrative instance featuring the root airfoil, offering a visualization of its equations and the associated fitting curve.

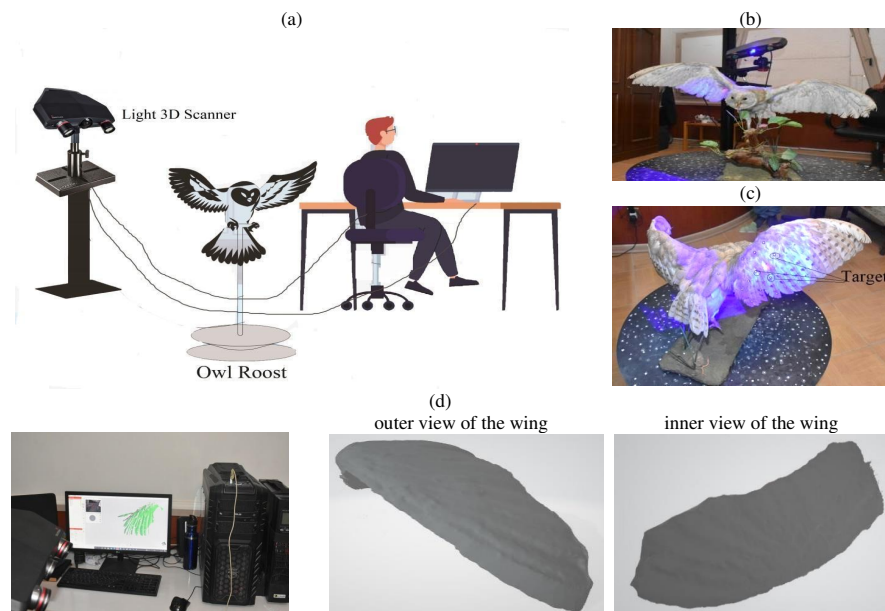


Figure 1: the experimental set-up. (a) A general illustration of the experimental setup. (b) An owl with the camera laser (c) A close view of the laser light sheet with targets on the owl wing (d) 3-D scan output of the owl's wing.



Figure 2: (a) Barn owl wing, (b) inverted owl wing model.

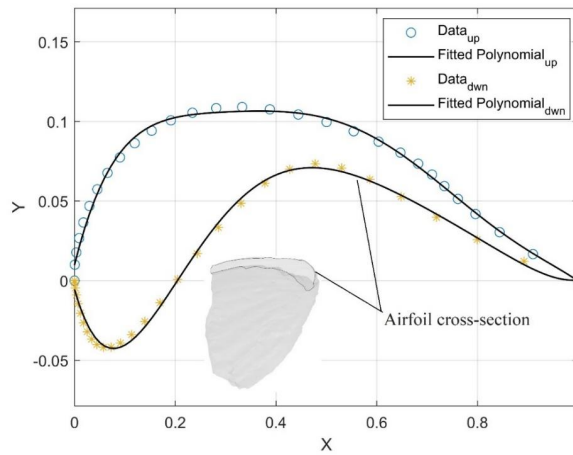


Figure 3: Fitting Results for Airfoil Profile.

The extracted airfoil exhibits a maximum thickness of 12% at  $x/c=0.11$ . In the study by Klan et al. [19], these characteristics are documented as 14.77% and 0.15, respectively. It's important to acknowledge a slight variance in the airfoil's overall specifications, potentially stemming from the owl's taxidermy process and water loss from the bird's body. Furthermore, the airfoil features a cusp-type trailing edge, consistent with the research of Ricks et al. [28], which suggests that such a slender trailing edge is associated with reduced noise generation. As indicated, the spatial arrangement of the upper and lower airfoil surfaces has been approximated using a 6<sup>th</sup> degree polynomial function. This function, denoted as  $y_c = ax^6 + bx^5 + cx^4 + dx^3 + ex^2 + fx + g$ , has coefficients detailed in Table 1. Here, 'x' represents the positional coordinates along the direction of the airfoil chord.

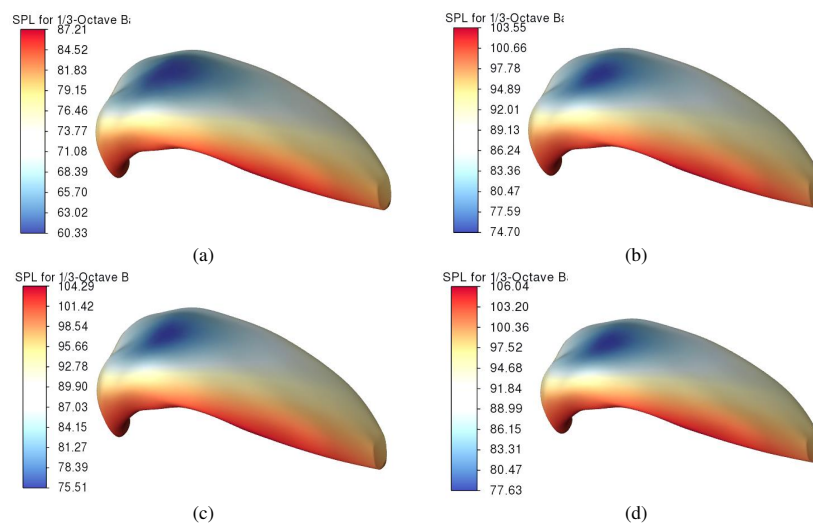
Table 1: Polynomial Coefficients for Geometric Approximation of Airfoil Surfaces.

Type	a	b	c	d	e	f	g
upper surface	-6.62	23	-30.77	20.1	-6.98	1.266	0.01
lower surface	7.76	-28.33	41.31	-29.72	10.10	-1.1	-0.0056



## 162 Blade Design

163 After scanning the owl's wing, as depicted in Figure 4, a comprehensive numerical analysis  
 164 was conducted in the vicinity of the owl's wing across a range of frequencies. The simulation  
 165 was carried out using ANSYS-FLUENT software, employing the LES turbulence model and  
 166 FW-H acoustic analogy. The domain is subjected to boundary conditions, with velocity inlet  
 167 and pressure outlet. Additionally, a symmetry condition is applied to surface included the wing  
 168 root. The computational domain had dimensions of  $0.5 * 0.8 * 2$  meters, with a mesh count of  
 169 0.5 million. Based on sound pressure level measurements conducted by Gruschka et al. [29], it  
 170 was established that the owl's sound remains inaudible beyond a 3-meter distance for  
 171 frequencies below 2000 Hz. Consequently, it was decided to incorporate SPL contour tuned to  
 172 frequencies of 500, 1000, 1600, and 2000 Hz. Moreover, a recurring pattern can be observed  
 173 across different frequencies in all four cases. Additionally, the following section presents the  
 174 OASPL curve for the wing sections along the span direction. Upon examination of this curve,  
 175 it became evident that specific locations along the span of the owl's wing consistently exhibited  
 176 lower noise levels compared to others. Consequently, these positions were selected as the  
 177 preferred locations for airfoil extraction. In the selection of these positions, careful  
 178 consideration was given to choose justifiable points within the range. These positions were  
 179 identified at 3%, 12.5%, 27%, 44%, and 68% of the owl wing's span, respectively. The  
 180 reduction in noise during owl flight was attributed to the distinctive characteristics of its airfoil.



181 Figure 4: Noise level contours (a) 500 Hz, (b) 1 kHz, (c) 1.6 kHz and (d) 2 kHz.

182 In light of this discovery, as depicted in Figure 5a, the decision was made to extract and employ  
 183 these airfoil characteristics from the owl's wing as the preferred airfoil profiles for blade design.  
 184 The distribution of OASPL shows minimal variations along the owl's wingspan. Interpolation  
 185 has been utilized to pinpoint cross-sectional data, with selections made at intervals of 7%, 15%,  
 186 30%, 50%, and 70% of the blade span. To align with the structural characteristics of the WT,



the airfoil for the WT is chosen based on approximately 70% of the actual owl wing (Arm wing). The selected positions of the airfoils are approximately consistent with the minimum locations in the OASPL curve. Consequently, the airfoil of the last section is selected as the tip airfoil for the wind turbine. Wolf and Konrath [30] conducted measurements of the three-dimensional shape of an owl's wing during a flapping cycle. The airfoils derived from the wings of the taxidermy owl examined in this study closely resemble the configuration observed during the gliding phase at a position approximately 5 meters along the right-to-left flight direction. Furthermore, as shown in Figure 5b, these acquired airfoil profiles were utilized to create various sections of the ducted wind turbine. These airfoils were subsequently proportionally scaled and adjusted according to the specifications outlined in Table 2 to align with the design requirements of the desired wind turbine. The ducted wind turbine is designed based on the owl's wing but with some modifications to improve the noise reduction for urban installations.

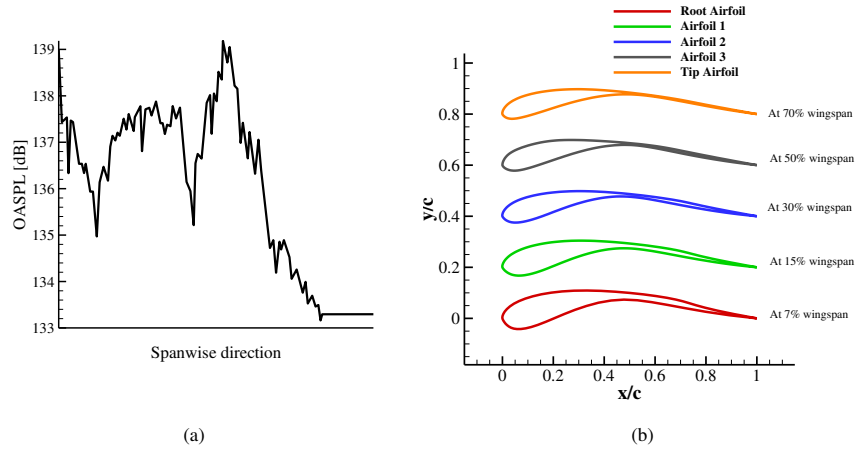


Figure 5: (a) the distribution curve of OASPL in the spanwise direction and illustrating the definition of the position of the selected airfoils, (b) Airfoils section inspired by owl wings.

As depicted in Figure 6, the airfoils for various sections draw inspiration from owl wings in shaping the desired geometry (refer to Table 2). Within the framework of DWT blade design, a consistent airfoil is established within each of the delineated sections. However, to seamlessly interlink these sections, the loft feature is implemented in CAD Software, allowing for adaptable variation of the airfoil along the span. The turbine has a duct length of 1 meter and an internal diameter of 1.6 meters.

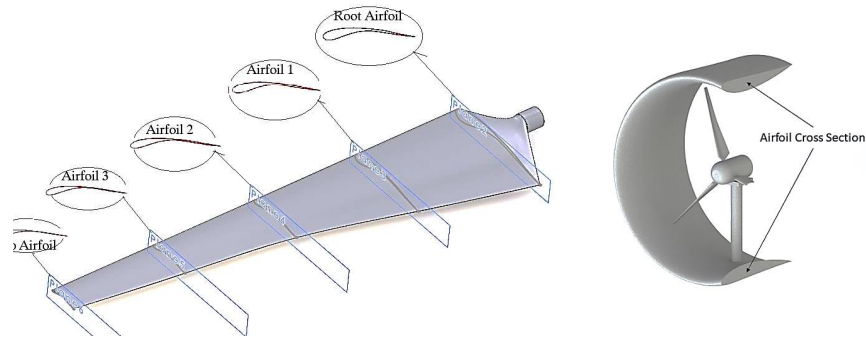


Figure 6: Schematic of the ducted wind turbine and multi-section blade.

Table 2: Design Features of Turbine.

Airfoils	Chord (m)
Root Airfoil	0.16
Airfoil 1	0.12
Airfoil 2	0.08
Airfoil 3	0.06
Tip Airfoil	0.04
Duct Airfoil	1

## Cases examined

This research involved two simulations, which are described in brief below. In this simulation, unsteady, compressible conditions are used in conjunction with LES turbulence model and FW-H acoustic analogy.

- Case 1: (Benchmark Blade- DonQi® wind turbine [31]).
- Case 2: (Multi-section Blade) Use of airfoils of different sections of the owl wing for blade sections: This case was done to investigate the effect of changing the airfoil and the use of airfoils of different sections of the owl wing in different sections of the blade.

## Governing equations

This study utilized computational fluid dynamics with the Large Eddy Simulation (LES) method for fluid flow analysis and the Ffowcs Williams and Hawkins (FW-H) acoustic analogy for acoustic analysis. The governing equations for each method are detailed below.

## Fluid dynamics

Turbulent flows are characterized by eddies with a wide range of length and time scales. In the Large Eddy Simulation (LES) method, large eddies are solved directly and small eddies are also modeled. The governing equations employed for LES are obtained by filtering the time-dependent Navier-Stokes equations in either Fourier (wave-number) space or configuration (physical) space. Filtering the Navier-Stokes equations, one obtains [32]:

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x_i} (\rho u_i) = 0 \quad (1)$$

229 And

$$\frac{\partial}{\partial t} (\rho u_i) + \frac{\partial}{\partial x_j} (\rho u_i u_j) = \frac{\partial}{\partial x_j} \left( \mu \frac{\partial \sigma_{ij}}{\partial x_j} \right) - \frac{\partial \bar{p}}{\partial x_i} - \frac{\partial \tau_{ij}}{\partial x_j} \quad (2)$$

230 where  $u$ ,  $\rho$ ,  $\mu$  and  $p$  are the fluid velocity, density, turbulent viscosity, and static pressure,  
 231 and  $i$  and  $j$  are the subscripts with 1 and 2 for the  $x$  and  $y$  directions. Also,  $\sigma_{ij}$  is the stress  
 232 tensor due to molecular viscosity and  $\tau_{ij}$  is the subgrid-scale stress.

### 233 Acoustics

234 Lighthill proposed the theory of the difference between real flow and reference flow and called  
 235 it analogy. Ffowcs Williams-Hawkins (FW-H) then challenged it and used to give solutions  
 236 to Lighthill's equation for a medium that includes moving surfaces and convected turbulent  
 237 flow. In this research, FW-H formulation has been used to model the propagation of sound  
 238 from a moving source [33, 34].

$$\begin{aligned} \frac{1}{c_\infty^2} \frac{\partial^2 p'}{\partial t^2} - \frac{\partial^2 p'}{\partial x_i^2} = \frac{\partial^2}{\partial x_i \partial x_j} \{ T_{ij} H(f) \} - \frac{\partial}{\partial x_i} \{ [P_{ij} n_j + \rho u_i (u_n - v_n)] \delta(f) \} \\ + \frac{\partial}{\partial t} \{ [\rho_\infty v_n + \rho (u_n - v_n)] \delta(f) \} \end{aligned} \quad (3)$$

239 Where  $u_n$  and  $u_i$  are the fluid velocity in the normal direction of the integration surface and  
 240 in  $x_i$  direction, respectively.  $v_n$  and  $v_i$  represent the normal velocity of the integration surface  
 241 and the surface velocity component in  $x_i$  direction.  $H(f)$  is Heaviside function and  $\delta(f)$  is  
 242 Dirac delta function.  $p'$  is sound pressure in the far field ( $p' = p - p_\infty$ ),  $n_j$  normal vector  
 243 pointing to the external area ( $f > 0$ ),  $c_\infty$  is speed of sound in the far field,  $P_{ij}$  is compressive  
 244 stress tensor and  $T_{ij}$  is the Lighthill's stress tensor, given by:

$$T_{ij} = \rho v_i v_j + p_{ij} - (\rho - \rho_\infty) c_\infty^2 \delta_{ij} \quad (4)$$

245 To solve Equation (3), the Green's function must be used to the open area. The complete  
 246 solution involves the calculation of surface and volume integrals, the first representing  
 247 monopole, dipole, and partially quadrupole acoustic sources, and the second representing  
 248 quadrupole sources in the area outside of the source surface. The volume integral becomes  
 249 negligible when the Mach number value of the flow is small and the source area covers the  
 250 source area. In Ansys Fluent, choosing a source on a solid surface-like rotor, the volume  
 251 integrals are neglected, and then the equation takes the following form [35]:

$$p'(\vec{x}, t) = p'_T(\vec{x}, t) + p'_L(\vec{x}, t) \quad (5)$$

In equation (5),  $t$  is the observer time,  $\vec{x}$  is the receiver position. The subscripts T and L refer to the thickness (monopole) and loading (dipole) components, respectively and are given as follows [33, 34]:

$$4\pi p'_T(\vec{x}, t) = \int_{f=0} \left[ \frac{\rho_\infty (\dot{U}_n + U_n)}{r(1-M_r)^2} \right] ds + \int_{f=0} \left[ \frac{\rho_\infty U_n (r\dot{M}_r + c_\infty (M_r - M^2))}{r^2(1-M_r)^3} \right] ds \quad (6)$$

$$4\pi p'_L(\vec{x}, t) = \frac{1}{c_\infty} \int_{f=0} \left[ \frac{\dot{L}_r}{r(1-M_r)^2} \right] ds + \int_{f=0} \left[ \frac{L_r - L_M}{r(1-M_r)^2} \right] ds + \frac{1}{c_\infty} \int_{f=0} \left[ \frac{L_r \left\{ r\dot{M}_r + c_\infty (\dot{M}_r - M^2) \right\}}{r^2(1-M_r)^3} \right] ds \quad (17)$$

Where:

$$U_i = v_i + \frac{\rho}{\rho_\infty} (u_i - v_i)$$

$$L_i = P_{ij} \hat{n}_j + \rho u_i (u_n - v_n)$$

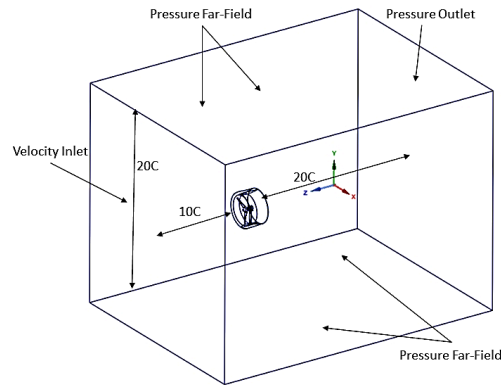
Where  $M$  and  $r$  represent the surface velocity vector and the unit radiation vector. The two terms  $p'_T(\vec{x}, t)$  and  $p'_L(\vec{x}, t)$  in Equation (5) are referred to as thickness and loading terms, respectively.

#### Numerical solver

The commercial software ANSYS-FLUENT has been used for all simulations carried out in this work. For an accurate calculation of the flow field around the blade, the LES solution was calculated with the Fluent SIMPLE solver. This approach has been proven suitable to describe similar problems in wind turbines. This solver used the finite volume method as a discretization procedure with Bounded Central Differences for momentum and Second Order Central Differences for pressure. A Bounded Second Order Implicit scheme is used for the time marching method in the present work as a temporal discretization scheme with the convergence criteria of  $10^{-4}$ . Since the Mach number at the blades of a wind turbine is always less than 0.2, the air has been modeled as incompressible for reducing the computational costs while maintaining accuracy.

271 The 3D mesh of the full rotor was carried out first by drawing it in the SpaceClaim software,  
272 while the mesh generation was performed using Fluent Meshing. For this problem, a  
273 polyhexcore mesh is generated. The computational domain is divided into two parts: the  
274 internal rotating field and the external relatively stationary flow field. The interfaces are set to  
275 transfer data between the rotational and stationary parts. Coupled problems between the two  
276 parts have a significant influence on the accuracy of numerical simulation. In the present study,  
277 the sliding mesh model is used to account for the rotation of the blades.

278 Figure 7 shows the boundary conditions, including velocity inlet and pressure outlet boundaries  
279 used to simulate the far field flow. In this case, the velocity of the free stream is 5 m/s,  
280 corresponding to the Reynolds number (Re) based on the duct chord length  $c$  ( $Re=3.4 \times 10^5$ ).  
281 The rotational speed of the wind turbine is 39.84 rad/s. distance from the main inlet boundary  
282 to the leading edge of the blade is  $10 \times c$  and the distance from the leading edge to the main  
283 outlet boundary is  $20 \times c$ .



284

285 Figure 7: Computational domain used for the LES simulation. The length is indicated in terms of duct chord  
286 length  $c$ .

287 According to Figure 8, the computational domain is discretized by about 6 million cells. The  
288 fine mesh is used on the whole rotor surface and gradually becomes coarser as the distance  
289 from the blades increases. Technique based on the previous results of k-epsilon turbulence  
290 model have been used to evaluate the grid resolution in LES method. In this technique,  
291 parameter  $f$ , has been calculated for the entire computing field by RANS model. The parameter  
292  $f$  denotes the ratio of the integral turbulence length scale to the filter width. This can be  
293 simplified and expressed as  $f = k^{3/2} / (\epsilon \times (\text{Cell Volume})^{1/3})$ , where  $k$  represents kinetic  
294 energy and  $\epsilon$  signifies turbulent dissipation [36]. In most areas, this value is of the order of 5-  
295 10, which indicates the good quality of the mesh. Therefore, the grid resolution meets the  
296 requirements for the LES calculation.

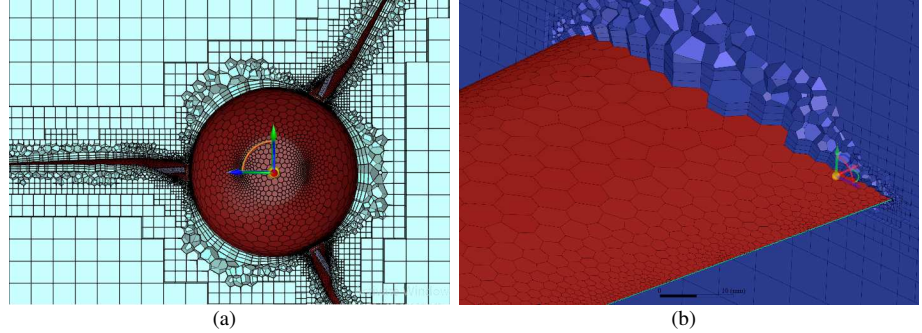


Figure 8: Schematic view of the computational domain. (a) Front view of DWT and (b) Side view of the blade.

## Results and Discussion

### Validation

The validation method for this research was conducted with great attention to details. It involved a rigorous comparison between the thrust force coefficient ( $C_T$ ), which is derived from the pressure coefficient ( $C_p$ ) along the chord, and the reference data obtained from Ten Hoopen's study described in reference [37]. The main objective was not to precisely reproduce the  $C_p$  values at each location on the surface along the chord, but rather to guarantee that the general pattern of  $C_p$  distribution closely corresponds to the reference, as illustrated in Figure 9. The key element of our validation process involved comparing the precise values of  $C_T$ , with a specific emphasis on the correlation between the thrust ratio in our simulation and the reference of [37].

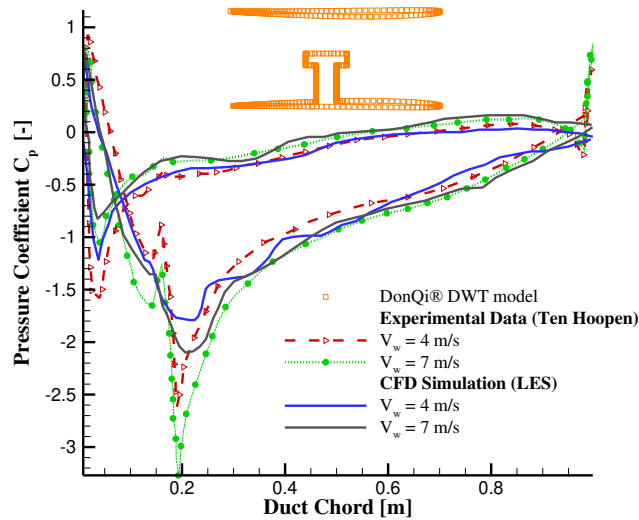
It is important to note that Ten Hoopen's work lacks detailed information regarding the manufacturing accuracy of the model and the precision of the measuring instruments. This implies that achieving a precise match for local  $C_p$  values between the simulated results and the reference data may not be possible. Hence, our validation focused on confirming the pattern of behavior and the distribution curve of pressure.

In the present context, the validation of the simulation, which relies on the Donqi blade results and is supplemented by the experimental findings of Ten Hoopen [37], demonstrates a fundamental similarity in pressure distributions. However, there are slight variations observed in the vicinity of the suction peak region, which may be attributed to assumptions made during the simulation or inherent uncertainties in the experimental data. This validation verifies that the aerodynamic characteristics of the ducted wind turbine (DWT) are accurately represented and simulated, especially when examining how the pressure distributions react to different wake propagation velocities ( $V_w$ ), which in turn affect the maximum absolute pressures on various sides of the airfoil. Therefore, the duct thrust force coefficient ( $C_T$ ) has played a crucial role in our work by calculating the grid resolution and assessing the aerodynamic changes in the DWT models. This ensures that our technique closely matches the established experimental standards. Table 3 presents a comparison of various grids against experimental and numerical

326 results. The table indicates that the error for the 6 million grid was kept below 10%, meeting  
 327 the required accuracy standards. Therefore, the chosen grid number for this study was 6  
 328 million. The multi-section blade exhibited a 6.4 % increase in thrust force coefficient compared  
 329 to the conventional blade.

330 Table 3: Comparison of the thrust force coefficient.

	Experiment al Value [37]	2.7 million numerical value (Benchmark )	LES Error (%)	4.3 million numerical value (Benchma rk)	LES Error (%)	6 million numerical value (Benchmar k)	LES Error (%)	6 million numeri cal value (Multi- section)	Difference Value (percentage increase)
thrust force coefficient, $C_T$	0.689	0.811	17.7	0.773	12.2	0.735	6.7	0.782	6.4



331  
 332 Figure 9: Comparison between numerical and experimental solution of pressure distribution.

333 To validate the acoustic findings in our present study, we have conducted a comparative  
 334 analysis of the Power Spectral Density graph of the acoustic pressure, focusing on the blade  
 335 passing frequency, as illustrated in Figure 10. This evaluation was undertaken at the  
 336 microphone location set at 90 degrees for the DonQi® DWT model, as examined by Dighe et  
 337 al, [31]. It's evident that minimal deviations exist for blade passing frequencies below 2, with  
 338 more pronounced discrepancies emerging at higher frequencies. Overall distribution has  
 339 similar characteristic which the 2<sup>nd</sup> harmonic is less pronounced, and the 4<sup>th</sup> harmonic stronger.  
 340 Importantly, our LES approach not only captures these variations but also effectively  
 341 characterizes the transitional trends.



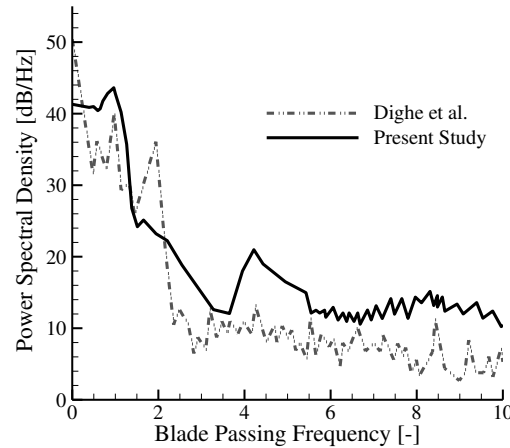
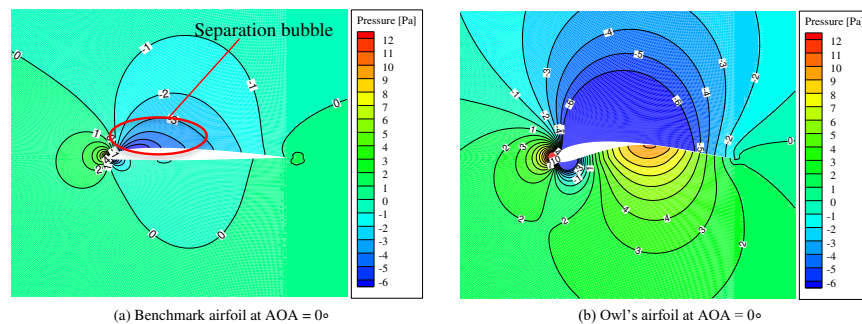


Figure 10: Validating Acoustic Results.

#### 344 Aerodynamics results

345 This section focuses on the analysis of the characteristic of the owl airfoil compared to a  
346 conventional airfoil in a 2D context used for wind turbine. The evaluation is carried out by  
347 analyzing the aerodynamic properties of both types of airfoils, and the results are discussed in  
348 this section.

349 Figure 11 illustrate the pressure contour for the owl and benchmark airfoils at angles of attack  
350 of 0 and 6 degrees. The unique shape of the owl airfoil, characterized by a thicker leading edge,  
351 higher curvature, and thinner trailing edge, generates a separation bubble. More pronounced in  
352 the owl airfoil Compared to a standard reference, leading to a more significant pressure  
353 difference and, in turn, increased lift (as depicted in Figure 12).



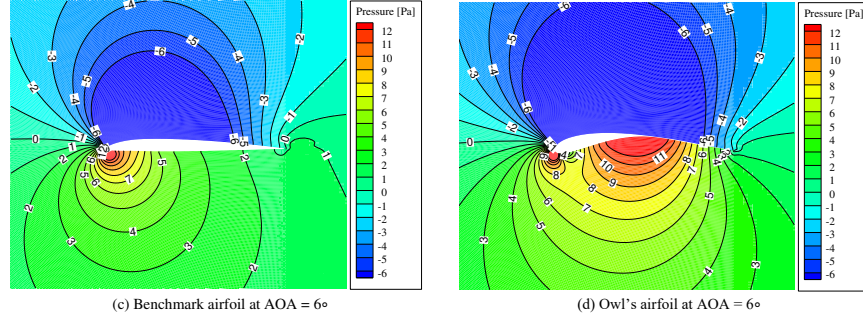


Figure 11: The pressure contour at different angles of attack.

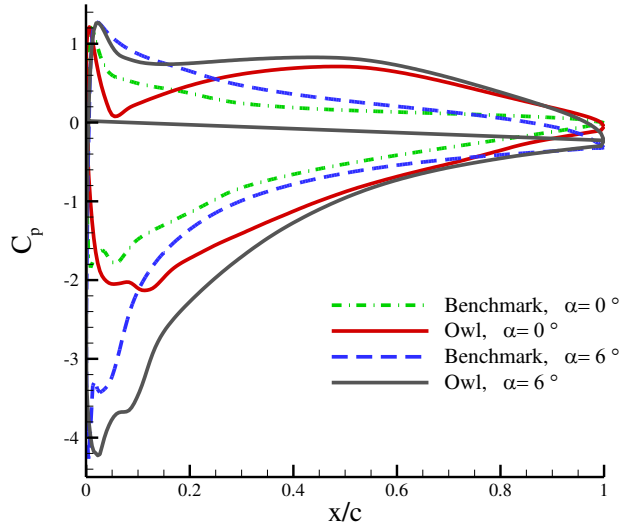


Figure 12:  $C_p$  comparison between Owl and benchmark airfoils for AOA= 0° and AOA= 6°.

In the owl airfoil, the blade's maximum camber, maximum thickness, and their respective positions are altered, enhanced lift performance, increasing the maximum lift coefficient from 1.8 to 2.8 and raising the stall angle of attack from 12° to 14° (as shown in Figure 13a). While the drag coefficient is higher for low attack angles in the owl airfoil, resulting in increased drag, this is a necessary tradeoff for generating more lift (as shown in Figure 13b). However, as shown in Figure 13c, the owl airfoil consistently outperforms the conventional airfoil in terms of lift to drag ratio all angles of attack. There is a noticeable improvement in the lift-to-drag ratio when the angle of attack reaches 5 degrees. Between angles of 8 to 10 degrees, the aerodynamic performance remains relatively stable. However, beyond this range, in all sections, there appears to be a decline in performance, possibly attributed to an increase in drag.

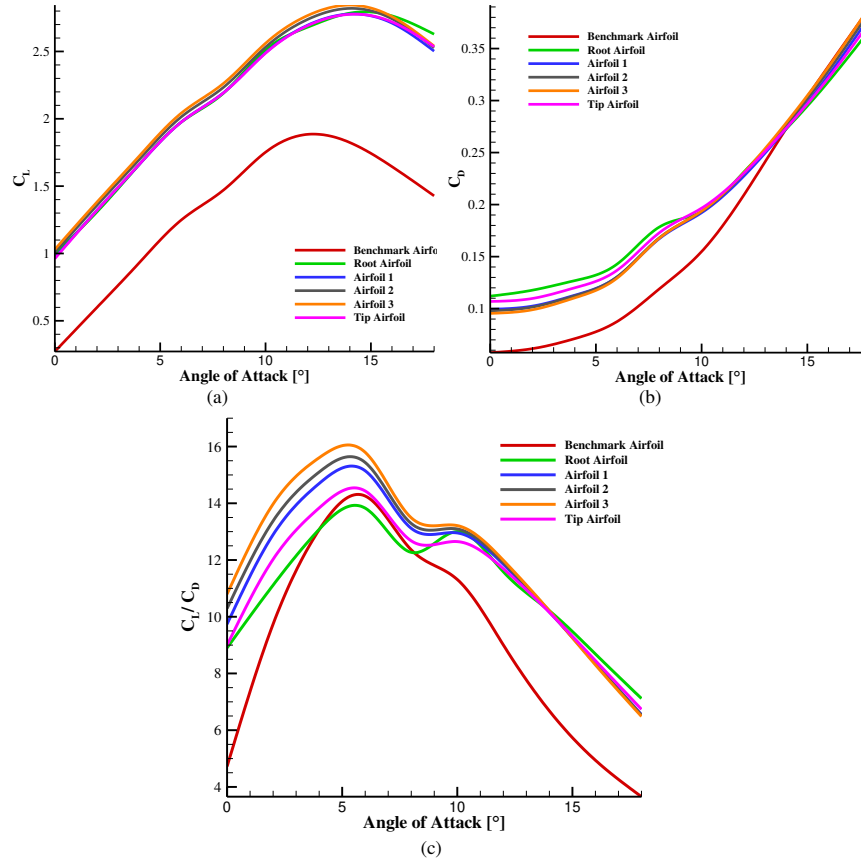


Figure 13: (a) Lift coefficients, (b) Drag coefficients and (c) Lift-to-drag ratio ( $C_l/C_d$ ) curves of the airfoils at different AOA.

This section explores and presents the aerodynamic features of the turbine after the integration of the owl airfoil. The implementation of the owl airfoil on the turbine is carried out, and the resultant aerodynamic properties are studied and analyzed.

To better illustrate the changes in the flow field and interactions between the turbine and the boundary layer as a result of the inlet flow, a 2D section of the flow velocity is shown in the XY plane in Figure 14a. As can be seen, there are areas of low speed around the duct and turbine holder. At the same time, there are also high-speed areas at the ends of the blades and downstream (in the wake) which is 25 m/s. This indicates that the duct acts as a diffuser and increases the speed of the incoming air.

For Case 2, the instantaneous flow fields around the ducted wind turbine are shown using the Q criterion in Figure 14b. The figure illustrates the formation of vortices on the inner wall of

the duct. Due to the increase in camber and changes in the thickness of the airfoil as the flow moves along the inner wall of the duct, the speed of the flow increases. The front vortices originate from the starting point of the inner side of the duct and adopt a helical shape as they get closer to the turbine. Consequently, this leads to an increase in turbulence structures and an increase in Turbulence Intensity (TI), which ultimately breaks up the larger front vortices into smaller ones. It also shows that the LES turbulence model predicts the turbulence fluctuations around a DWT well and provides a clearer picture of the complex flows around the turbine.

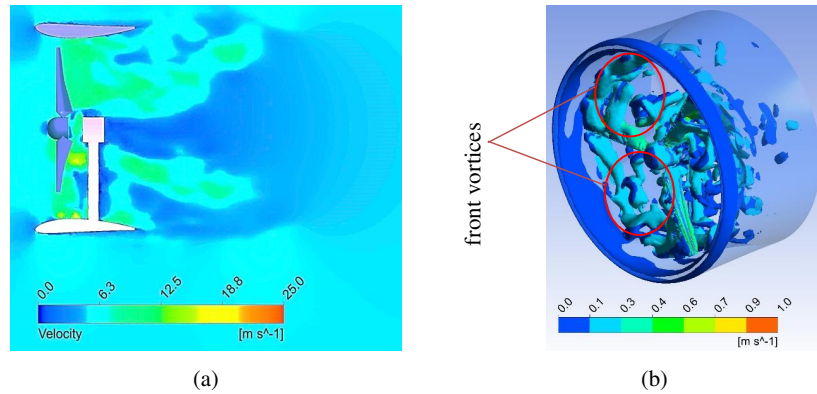


Figure 14: (a) Velocity contour, (b) Q criterion.

In the context of a wind turbine, the conversion of mechanical energy into electrical energy hinges on the performance of the aerodynamic system, quantified as the power coefficient ( $C_{PWR}$ ). In parallel, an essential factor in the design of turbine blades is the Tip Speed Ratio (TSR), representing the ratio between the linear speed of the blade tip and the wind speed. Consequently, this section delves into the variations in the power coefficient concerning TSR for both benchmark and multi-section blades. As illustrated in Figure 15,  $C_{PWR}$  exhibits a non-linear relationship with TSR, with the maximum  $C_{PWR}$  value occurring at  $TSR = 8$  in both cases. This maximum value signifies the peak efficiency of the DWT. Thus, the selection of the optimal TSR value is of paramount importance. Furthermore, as depicted in the figure 15, for TSR values below 5, there is negligible discrepancy in  $C_{PWR}$  between the two blade types, with their performance differing by less than 10%. However, as TSR values increase, the  $C_{PWR}$  parameter demonstrates a significant rise, underscoring the benefit of employing multi-section blades.

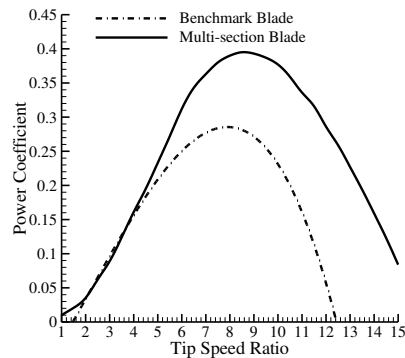


Figure 15: Variation in Power Coefficients for Benchmark and Multi-Section Blades across Various Tip Speed Ratio.

#### Aeroacoustics results

To investigate the sound around the turbine, according to Figure 16, thirty receivers are being used in the current study. The receivers are positioned at a distance of 1.5 times the chord length of the Duct airfoil. Receivers are spaced 12 degrees apart.

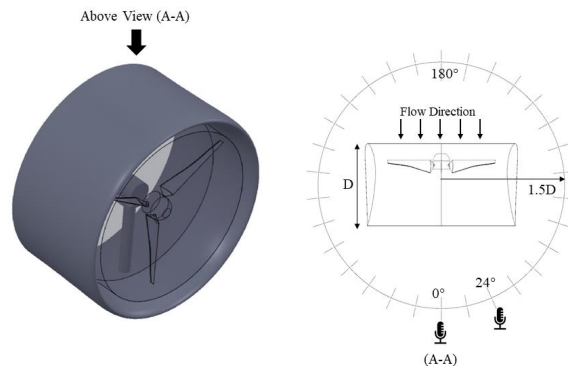
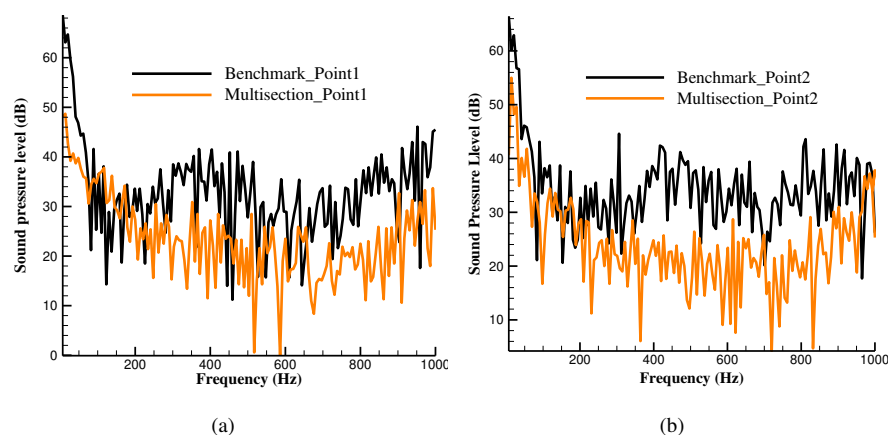


Figure 16: Receivers position.

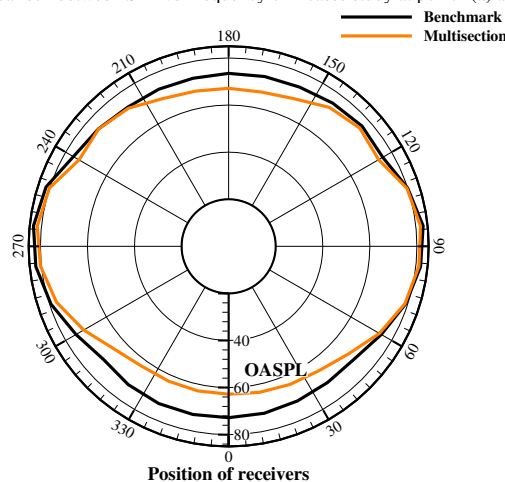
Based on the receiver position curve, two specific points are analyzed in this study, point 1 at 0 degrees and point 2 at 24 degrees. Figure 17 displays sound pressure level graphs for both points, showing the frequency response for two cases. By implementing owl wing airfoils, noise generated by free stream turbulence and trailing edge noise is reduced. Notably, this improvement has the most significant impact on higher frequencies. The multi-section results suggest that owl-inspired airfoils have proven effective in mitigating noise.

According to the polar curve of Overall Sound Pressure Level (OASPL), as shown in Figure 18 based on the position of the receivers, it can be seen that the use of airfoils of owl-inspired reduces the sound by an average of 6-8 decibels at 60 to -60 degrees (which represents the flow

419 exit area from the duct). As can be seen, in some positions, including 90 and 270 degrees, there  
420 is no change in the OASPL curve for owl-shaped wing (multi-section blade). It does not reduce  
421 sound in the radial direction. As a result, to reduce sound, it is recommended to make changes  
422 such as using perforated plates in the structure of the duct and its surfaces, as well as adding  
423 sound-absorbing materials inside.



424 Figure 17: Comparison between SPL vs Frequency of 2 cases study at point1 (a) and point2 (b).



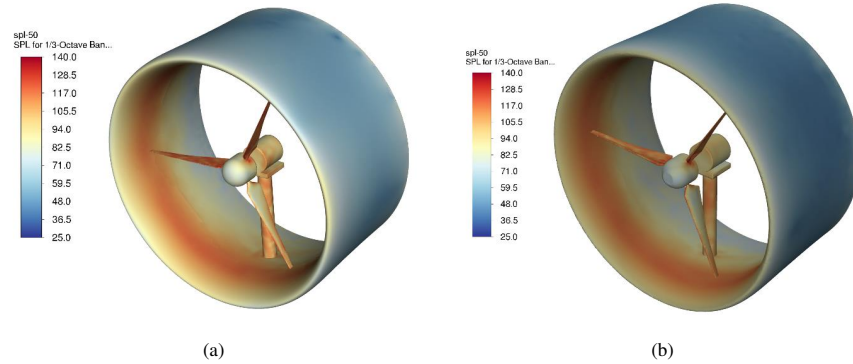
425  
426 Figure 18: Comparison between SPL of 2 cases study around the ducted.

427 As can be seen in Figure 19a, the highest sound pressure level occurs on the inner surface of  
428 the duct which the lowest level is on the outer surfaces of the duct. The highest SPL value on  
429 the inner surface of the duct is typically associated with the airfoil's thickest portion located at  
430 the leading edge. As a result, the use of a duct as a diffuser cover significantly reduces the  
431 sound and act as a barrier, as well as protect the surrounding environment from damage. Based

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432 on the contour presented in figure 19b, it appears that altering the blade foils did not result in  
433 any detrimental impact on the level of noise produced at the tip of the turbine duct. Also, figures  
434 19 illustrates how the surface under the airfoil, which is the suction area, has a more significant  
435 effect on noise production. An explanation for this can be found in the change in the camber  
436 line and the changes in the thickness of the airfoil. Therefore, it leads to an increase in  
437 turbulence structures and pressure fluctuations, resulting in more sound being produced on the  
438 lower surface of the airfoil. With comparing Figures 19a and 19b, it becomes apparent that the  
439 SPL value has decreased both downstream and on the inner surface of the duct. Further analysis  
440 can be conducted by referring to Figure 19c, which illustrates sound pressure levels specifically  
441 along the inner surface of the duct. In this segment, a section is formed on the interior of the  
442 duct. It is worth noting that modifications made to the turbine blade have a noticeable effect on  
443 the noise generated downstream, leading to a reduction in noise at the trailing edge of the inner  
444 duct. This noise reduction is approximately equal to 10 dB in all positions after the turbine  
445 tower. Intriguingly, the use of multi-section blades does not impact the noise levels in front of  
446 the rotor.  
447





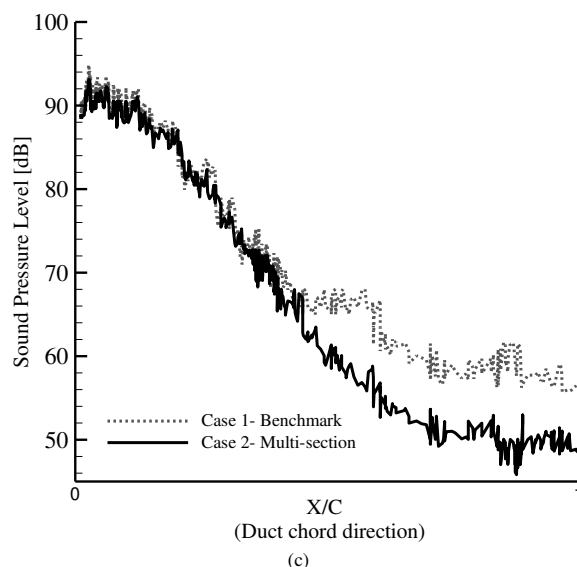
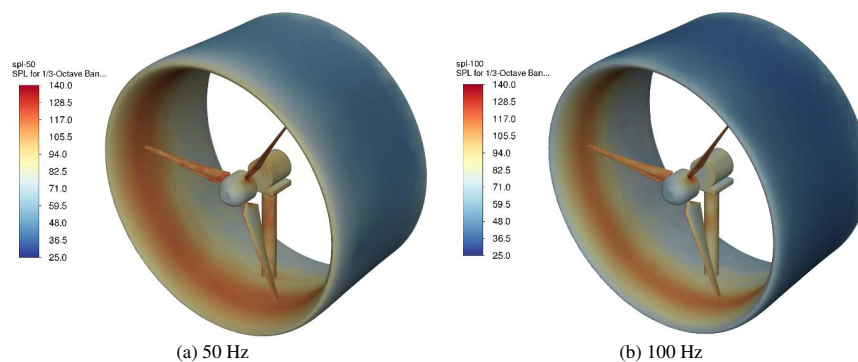


Figure 19: Noise level contour for frequency 50 Hz for cases number 1 (a) and 2 (b) and (c) Comparative Analysis of Sound Pressure Levels along the Inner Duct Surface between Cases 1 and 2.

Upon analyzing the frequencies ranging from 50 Hz to 500 Hz in case number 2 as shown in Figure 20, it becomes apparent that noise levels decrease as the frequency increases. The maximum noise production within the duct occurs in the regions near the leading edge to the tower, with the highest levels of noise generated in the corresponding areas of the blade tips. Also, this figure indicates that the most effective place to reduce noise is this area, which is the logical point to consider the perforated plate (punch) along with the absorber at the position of the inner junction to the turbine tower to the duct.



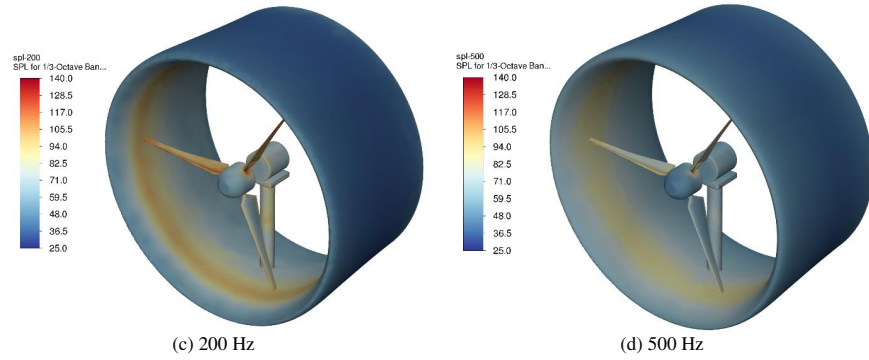


Figure 20: Noise level contour for frequencies 50 Hz to 500 Hz for case number 2.

## Conclusions

In this comprehensive study, we have employed advanced Large Eddy Simulation (LES) turbulence modeling and Ffowcs Williams-Hawkins (FW-H) acoustic analogy to meticulously analyze the aeroacoustic and aerodynamic properties of ducted wind turbine blades. These blades are innovatively designed, drawing inspiration from the silent flight mechanism of owl wings. Our primary aim was to address the critical challenge of reducing aerodynamic noise, particularly the noise generated by inflow turbulence and trailing edges, while concurrently ensuring that the aerodynamic performance remains uncompromised. The investigation revealed that the incorporation of bio-inspired airfoil sections, meticulously derived from owl wings through 3D scanning techniques, significantly influences the noise reduction and aerodynamic efficiency of wind turbines. Specifically, the application of these uniquely designed airfoils resulted in a notable reduction of aerodynamic noise by approximately 8 decibels, which translates to an improvement of about 12% when compared to conventional designs. This achievement underscores the potential of bio-inspired modifications in enhancing the environmental compatibility of wind turbines, particularly in urban settings where noise pollution is a significant concern. Moreover, our results demonstrated a 6.4% increase in the thrust force coefficient for the multi-section blade compared to the conventional blade design. The power coefficient analysis further revealed that the maximum power coefficient occurred at a Tip Speed Ratio (TSR) of 8 for both the benchmark and multi-section blades, emphasizing the optimized aerodynamic efficiency achieved through the bio-inspired design. Furthermore, our study delved into the aeroacoustic performance, where the findings indicated a substantial noise reduction in specific areas around the duct, especially downstream and on the inner surface. These results suggest that the strategic implementation of owl-inspired airfoil profiles not only benefits the noise profile but also contributes to the overall aerodynamic efficiency of the turbine. However, it's worth noting that the radial sound suppression remained unaffected, suggesting additional avenues for enhancing the aeroacoustic performance, possibly through further structural modifications or the integration of sound-absorbing materials. In conclusion, this research not only contributes valuable insights into the aeroacoustic and aerodynamic optimization of wind turbines but also highlights the potential of bio-inspired designs in the field of renewable energy. The significant reduction in noise levels, coupled with the maintenance of aerodynamic performance, presents a compelling case for the adoption of

such innovative design strategies in future wind turbine development, particularly in noise-sensitive environments. Our findings lay a solid foundation for future studies and the practical application of bio-inspired designs in enhancing the sustainability and community acceptance of wind energy solutions.

#### Data Availability

The data used to support the findings of this study are available from the corresponding author upon request.

#### Conflicts of Interest

The authors declare that there is no conflict of interest regarding the publication of this paper.

#### Funding Statement

The authors have no relevant financial or non-financial interests to disclose.

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